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Prevention of Significant Deterioration Greenhouse Gas Permit Application

Sand Hill Energy Center

Del Valle, Travis County, Texas

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Prepared For The City of Austin dba Austin Energy

Submitted to:

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TRC Environmental Corporation | The City of Austin dba Austin Energy Final

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List of Acronyms

AVO audio/visual/olfactory

BACT Best Available Control Technology

Btu British thermal unit

CAAA Clean Air Act Amendments
CCS carbon capture and storage
CFR Code of Federal Regulations

CO carbon monoxide
CO₂ carbon dioxide

CTG combustion turbine generator

DB duct burner

DOE US Department of Energy

DLN dry low-NO_x F Fahrenheit

FIP Federal Implementation Plan

Ft feet

GE General Electric
GHG greenhouse gas

GWP global warming potential

H₂O water

HAP Hazardous Air Pollutant

HHV higher heating value

HRSG heat recovery steam generator

HP high pressure

IP intermediate pressure

K degrees on the Kelvin scale

kW kilowatt

LAER Lowest Achievable Emission Rate

lb/hr pounds per hour

lb/MMBtu pounds per million British thermal units

LHV lower heating value

LP low pressure

MACT Maximum Achievable Control Technology

MMBtu/hr million British thermal units per hour

MW Megawatt

NETL National Energy Technology Laboratory

NO_x nitrogen oxides

NSPS New Source Performance Standards

NSR New Source Review

NTIS National Institute of Standards and Technology

O₂ oxygen

ppmvd parts per million by volume, dry basis

PNG pipeline natural gas

PSD Prevention of Significant Deterioration

PTE potential to emit

RACT Reasonably Available Control Technology

RBLC RACT/BACT/LAER Clearinghouse

SAR South Austin Regional
SHEC Sand Hill Energy Center

scf standard cubic feet

SCR Selective Catalytic Reduction

SER Significant Emission Rate

SIP State Implementation Plan

STG steam turbine generator

TCEQ Texas Commission on Environmental Quality

Tpy tons per year

USEPA United States Environmental Protection Agency

VOC volatile organic compounds

Executive Summary

The City of Austin (dba Austin Energy) is proposing to build-out the Sand Hill Energy Center (SHEC) located in Del Valle, Texas by adding a new pipeline natural gas (PNG) fired combustion turbine generator (CTG) and heat recovery steam generator (HRSG) with natural gas fired duct burners to the existing combined cycle electricity generating unit at SHEC. The new unit will share the existing 189 MW steam turbine generator (STG) with the combustion turbine and HRSG associated with existing combined cycle unit. The proposed new combustion turbine generator is an updated version of the same General Electric (GE) Model 7FA unit currently in operation as part of the combined cycle unit.

Because the SHEC facility is an existing major source of greenhouse gas (GHG) emissions and the project increases facility GHG emissions by more than 75,000 tons per year on a carbon dioxide equivalent (CO2e) basis, the project is also subject to prevention of significant deterioration (PSD) new source review program for GHG emissions. EPA is authorized under a Federal Implementation Plan (FIP) to issue PSD permits in Texas for GHG sources until Texas submits the required SIP revision for GHG permitting and it is approved by EPA. The key element of PSD review for GHG is the best available control technology (BACT) analysis.

Sources of GHG emissions include the combustion turbine and duct burner combustion emissions of carbon dioxide (CO₂), nitrous oxide (N₂O) and methane (CH₄), the natural gas piping component leaks and other releases of natural gas (including CH₄) and the electric circuit breaker sulfur hexafluoride (SF₆) leak emissions. The latter two sources are minimal compared to the combined cycle emissions and BACT is addressed for these with a combination of effective design and work practices.

The BACT analysis for the proposed new combined cycle unit's GHG emissions evaluates two applicable options: carbon capture and sequestration (CCS) and energy efficiency measures. As with other recent GHG BACT analyses for permits for similar units, it was determined that the CCS technology currently has significant technical hurdles but may also rule out based on poor cost-effectiveness, with the capital cost of carbon capture, transport and storage exceeding the capital cost of the proposed project. As such, electrical generation efficiency measures are proposed as BACT for the project GHG emissions. This BACT technology determination is supported based on the low-carbon pipeline natural gas fuel, and the very efficient combustion turbine combined cycle technology, which compares favorably to similar units.

Consistent with recent GHG PSD permits for natural gas-fired combined cycle projects, the proposed BACT emission limits include and output-based GHG limit (ton CO2e/MWh), a heat rate limit (Btu/kWh) and an overall GHG emissions cap that takes into account start-up, partload and duct burner emissions. Because of facility-specific design considerations, specifically the proposed CTG, HRSG and duct burner sharing an STG with another CTG, HRSG and duct burner that are not subject to GHG BACT, the proposed GHG emission limits must take a slightly different form that those for other brand new 2x1 combined cycle units. A typical combined cycle unit that is designed and built with one or more CTGs and STGs is amenable to a limit that includes the electric generation output of both the CTGs and STGs, in order to "take credit" for the excellent efficiency associated with this combined cycle technology and show very low GHG ton/MWh limits and Btu/kWh heat rate limits. Because of the shared STG in this "2 on 1" configuration, it is not possible to determine how much of the output generated by the steam turbine is attributable to the new CTG and HRSG versus the existing CTG and HRSG. As such, Austin Energy is proposing that the new unit's GHG limits be based on the output of the combustion turbine alone. As such, the proposed limits are greater in terms of the proposed ton/MWh and Btu/kWh limits, but are shown to be comparable to GHG BACT combined cycle limits (based on CTG and STG output) in recently issued permits.

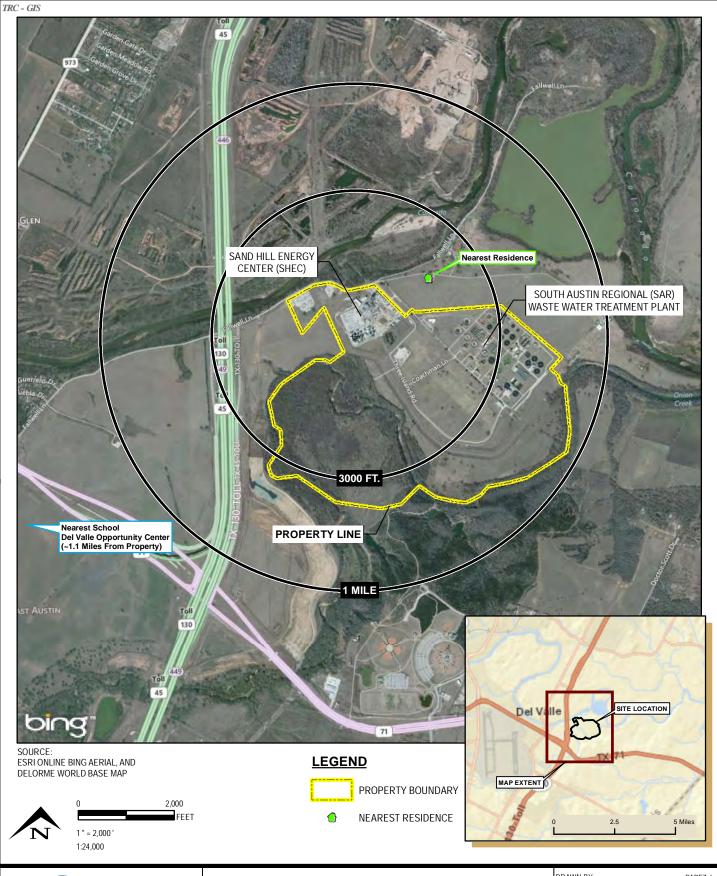
Section 1 Introduction

The City of Austin (dba Austin Energy) is proposing to build-out the Sand Hill Energy Center (SHEC) located in Del Valle, Travis County, Texas by adding to the existing combined cycle unit at the facility. The existing combined cycle unit at the SHEC was conceived and constructed to include this new unit when Austin's energy demands grew to the point where additional generating capacity would be required. The proposed project will add a new pipeline natural gas (PNG) fired combustion turbine and heat recovery steam generator (HRSG) to the existing combined cycle electricity generating unit at SHEC. A site location map is included as Figure 1-1 and a plot plan of the existing and proposed facility is presented as Figure 1-2.

Construction will include the installation of a General Electric (GE) model 7FA.04 combustion turbine and a heat recovery steam generator (HRSG) with natural gas fired duct burners (the Project). The new combustion turbine generator (CTG) is rated at 187 MW at International Standard Organization conditions. The new combined cycle unit will share an existing 189 MW steam turbine generator (STG) which is part of the existing combined cycle unit. Proposed emission controls technology includes dry low-NOx (DLN) combustion and selective catalytic reduction (SCR) for nitrogen oxides (NO_x) emission control and an oxidation catalyst to reduce emissions of carbon monoxide (CO) and volatile organic compounds (VOC).

The City of Austin will submit an amendment application to the Texas Commission on Environmental Quality (TCEQ) to authorize the addition of this second combustion turbine and HRSG at its SHEC facility. On June 3, 2010, the U.S. Environmental Protection Agency (EPA) published final rules for permitting sources of greenhouse gases (GHGs) under the prevention of significant deterioration (PSD) and Title V air permitting programs, known as the GHG Tailoring Rule.¹ After July 1, 2011, new sources having the potential to emit more than 100,000 tons per year of GHGs and modifications increasing GHG emissions more than 75,000 tons per year on a carbon dioxide equivalent (CO2e) basis at existing major sources are subject to GHG PSD review, regardless of whether PSD is triggered for other pollutants. The existing SHEC facility is an existing PSD major source based on potential criteria pollutant emissions greater than 250 tons per year and GHG emissions greater than 100,000 tons per year of CO2e.

¹ 75 FR 31514 (June 3, 2010)

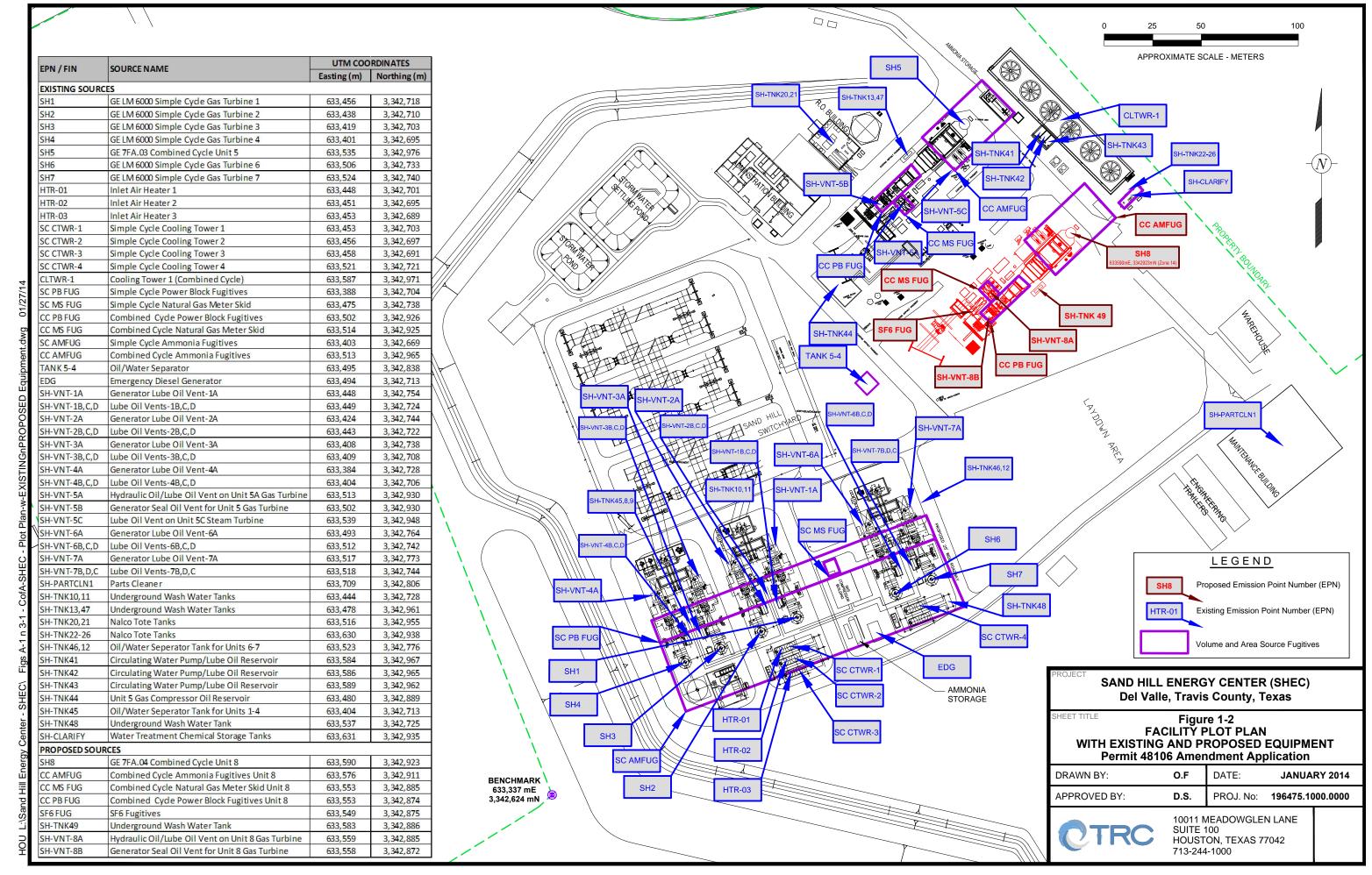




10011 Meadowglen Lane Houston, TX 77042 Phone: 713.244.1000 SAND HILL ENERGY CENTER
DEL VALLE, TRAVIS COUNTY, TEXAS

FIGURE 1-1 SITE LOCATION MAP

DRAWN BY:	PAPEZ J
APPROVED BY:	STANKO E
PROJECT NO:	196475-001-002
FILE NO.	196475-002am1.mxd
DATE:	AUGUST 2013



On December 23, 2010, EPA issued a Federal Implementation Plan (FIP) authorizing EPA to issue PSD permits in Texas for GHG sources until Texas submits the required SIP revision for GHG permitting and it is approved by EPA.²

The SHEC proposed Project for the addition of a second combined cycle combustion turbine triggers PSD review for GHG regulated pollutants because the project will increase GHG emissions by more than 75,000 tons/year and the site is an existing major PSD source. This application is being submitted to EPA pursuant to its regulatory authority under the FIP. Included in this application is a project description including a description of current operations, GHG emissions calculations, a PSD applicability determination and a GHG Best Available Control Technology (BACT) analysis. TCEQ permit application forms, emission calculations, and BACT analysis documentation are appended to this document.

² 75 FR 81874 (December 29, 2010)

Section 2 Project Description

2.1 Need for the Facility and Conceptual Design

Austin Energy requires additional generation to support a fast growing population and job growth in both the City of Austin and Travis County. Austin has been the fastest growing city in the country for the past three years. Since 2009 the population of Travis County has increased by 98,415 individuals, an increase of almost 10%³. Current projections indicate that county population is expected to continue to increase at this rate adding another 102,000 people by 2017⁴. Travis County has added 67,186 new jobs since 2009⁵ and this trend is expected to continue and keep pace with the projected population growth. The existing STG at the SHEC was sized to allow for population growth and increased power demands by accommodating the installation of an additional combustion turbine and HRSG.

Annual residential electricity consumption in Texas for 2011 was 145,654,228 MWh, an increase of 15,857,077 MWh (or 10.9%) from just two years earlier⁶. The population of the state increased by 892,379 individuals over this same two-year period, growing from 24,782,302 to 25,674,681 as of July 1, 2011 ⁷.

Based on the current average residential electricity usage per person of 5.673 MWh/yr/person for Texas, and a projected population increase in Travis County of over 200,000 persons from 2009 to 2017, the residential electricity demand is projected to increase by 1,136,968 MWh/year. The maximum additional capacity of the new unit is approximately 206 MW and this translates to 1,443,648 MWh annually based on a capacity factor of 80%. Therefore, the projected increase in local residential demand alone (over an eight-year period) represents 79% of the additional power available from the project.

2.2 The Existing SHEC Facility

The existing facility equipment, operations and emissions are regulated under Prevention of Significant Deterioration (PSD) permit No PSDTX1012M1 and Texas Commission on Environmental Quality (TCEQ) Permit No. 48106. The current generating units include six natural gas fired GE LM6000 aero derivative design simple cycle combustion turbines and the

³ Perryman Group

⁴ Population estimates from Perryman Group

⁵ Travis County employment from the Texas Workforce Commission

⁶ Energy Information Administration

⁷ US Census

existing natural gas fired GE Frame 7FA combustion turbine combined cycle unit including natural gas fired duct burners, a HRSG and a steam turbine generator. The six simple cycle units are designated in the permit as EPN's SH1, SH2, SH3, SH4, SH6 and SH7. The first four units (SH1-4) commenced operation in 2001 and the two newer units (SH6 and SH7) commenced operation in 2010. These units have a nominal output rating of 50 MW each and serve as "peaking" units that start up to help meet demand during peak (higher) periods. The LM6000 turbines utilize GE's spray inter-cooled turbine (Sprint) design and power augmentation and include water injection and SCR for NOx control.

The existing combined cycle unit commenced operation in 2004 and is designated in the TCEQ PSD permit as EPN SH5 and has a GE 7FA.03 combustion turbine – a previous version of the 7FA model. The turbine is equipped with dry low NOx (DLN) (model DLN2.6) combustors. Its HRSG is equipped with natural gas fired duct burners and SCR. The steam turbine generator for this unit was sized to accommodate the addition of a second similarly sized combustion turbine, with a space immediately adjacent to the southeast of the SH5 unit for the proposed SH8 unit, as shown in the plot plan in Figure 1-2. The current combined cycle unit is a one-on-one (1 x 1) configuration (one CTG with HRSG and one STG), but following the addition of the proposed new turbine and HRSG it will be a 2 x 1 configuration (two CTGs/HRSGs and one STG). The present combustion turbine has nominal rated output of 164 MW and the steam turbine generator currently produces up to 157 MW but will be capable of up to 189 MW output with the addition of the proposed second combustion turbine. As such, the maximum combined generating output of the combined cycle unit will increase from 321 MW for the existing 1 x 1 configuration to 553 MW for the proposed 2 x 1 configuration. The STG was originally sized for the planned build-out to a 2 x 1 configuration.

The existing cooling tower was sized for the full STG capacity in the 2 x 1 configuration, so no new cooling tower capacity is needed. Saturated steam from the STG is condensed prior to being recirculated along with makeup water to the HRSG for reheating. Condenser cooling is provided by circulating water that is in turn cooled by ambient air in the direct-contact mechanical draft cooling tower. The water that is used in the cooling tower makeup is either potable water or reclaimed water and river water from the Colorado River that is treated onsite. The reclaimed water is obtained from the adjoining South Austin Regional (SAR) wastewater treatment plant.

Ancillary equipment includes two existing aqueous ammonia storage tanks (19% aqueous ammonia solution) that store the SCR reagent for the units. One aqueous ammonia tank stores SCR reagent for all six simple cycle turbines. The other tank stores ammonia solution for the combined cycle unit and would also serve the proposed new unit. The aqueous ammonia goes to a vaporizer unit and is then injected into the flue gas upstream of the SCR catalyst. There are

also four existing cooling towers and three natural gas fired inlet air heaters associated with the simple cycle units and one existing cooling tower associated with the combined cycle unit.

2.3 The Proposed Project

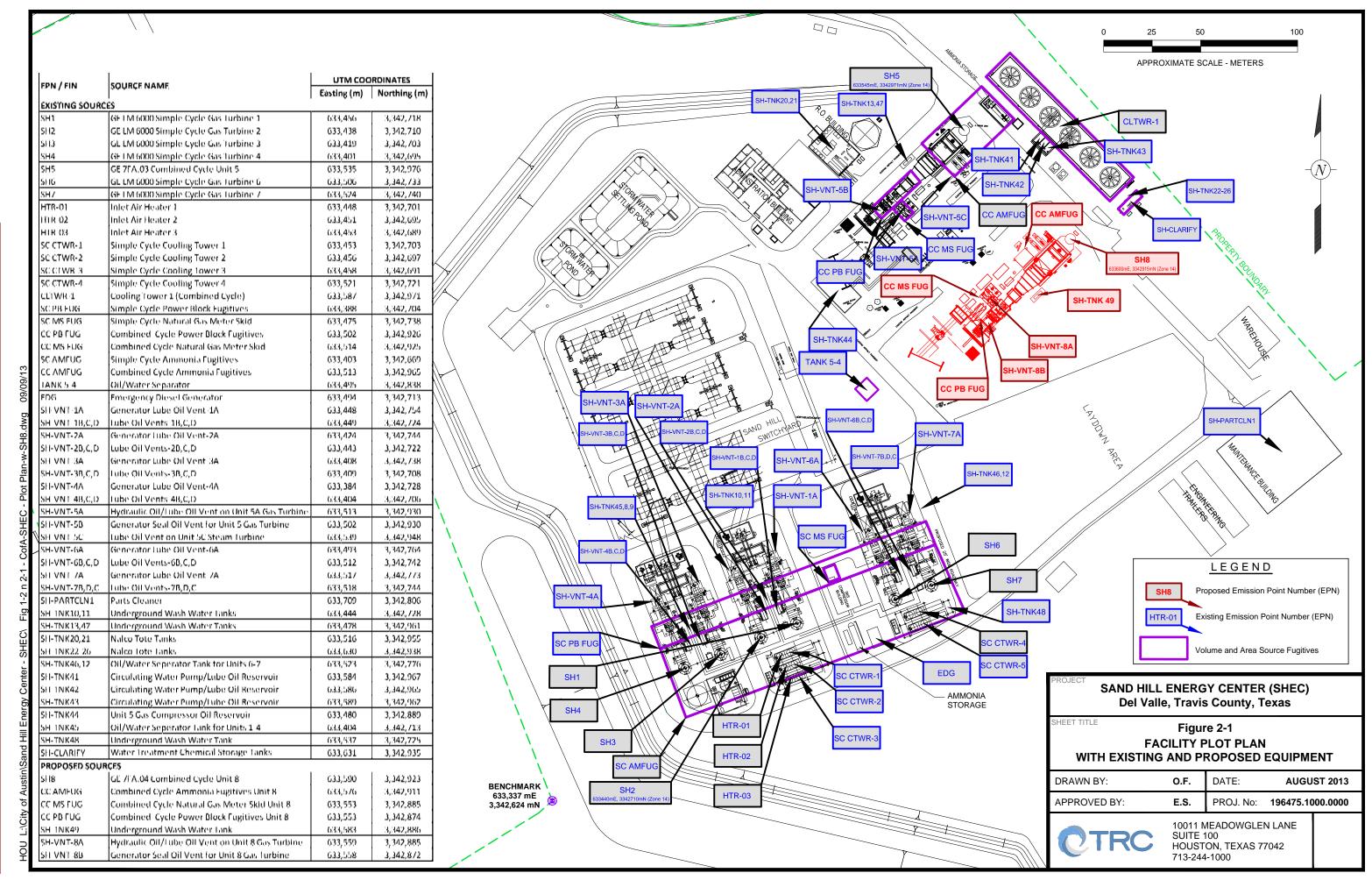
The new combined cycle unit is anticipated to operate as a base-loaded unit, with up to 8,760 full-load hours per year, but may also operate at partial loads, and/or start-up and shutdown as needed to meet electricity demand. The duct burners for the new unit will be rated at 681.5 MMBtu/hr based on the higher heating value (HHV) of the pipeline natural gas fuel, and may operate at full capacity for up to 8760 hours per year. The new combined cycle turbine is expected to start-up numerous times per year.

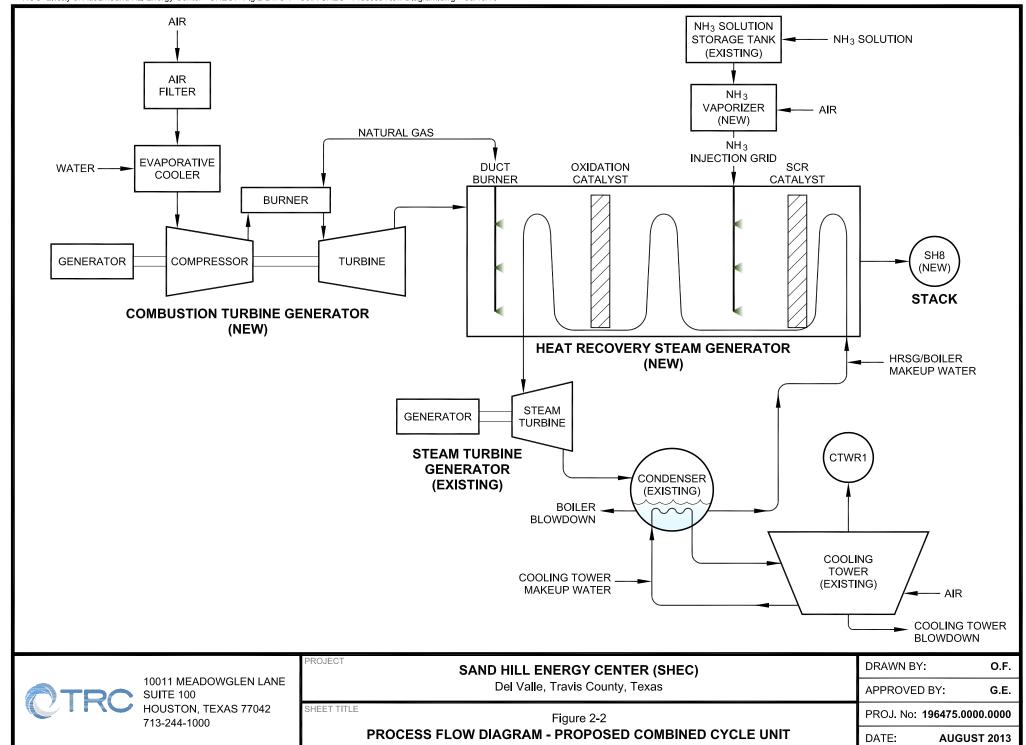
As described above, the new combustion turbine and HRSG will be located alongside the existing GE 7FA.03 turbine that is presently operating in combined cycle mode in a 1×1 configuration with a single CTG/HRSG supplying steam to a single STG. The existing STG is sized such that it will be able to accommodate the build-out with additional steam from the new HRSG of the proposed GE 7FA.04 combustion turbine; thus the new configuration will be 2×1 with two CTGs/HRSGs supplying steam to one STG.

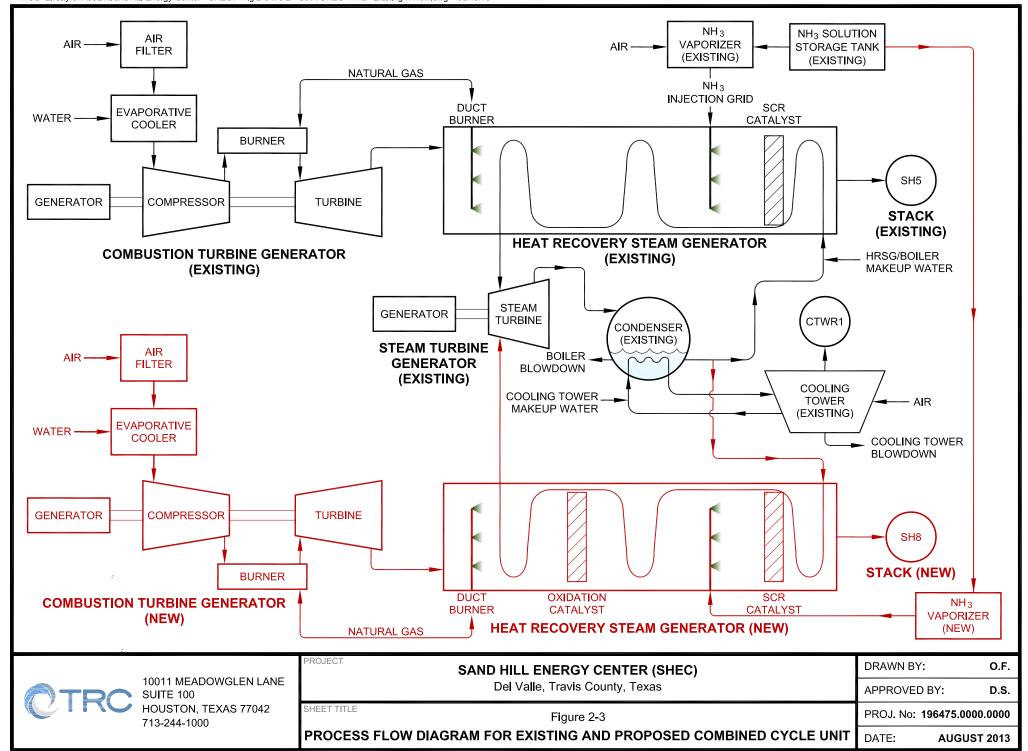
The proposed combustion turbine will utilize DLN combustors and SCR to control NOx emissions. Aqueous ammonia from the existing combined cycle ammonia storage tank will be vaporized in a new ammonia vaporizer dedicated to the SCR for the proposed unit. The proposed PNG-fired duct burner will have a maximum heat input capacity of 681.5 MMBtu/hr (HHV). An oxidation catalyst will be located in the HRSG downstream of the duct burners and upstream of the SCR ammonia injection grid and will control emissions of CO as well as VOC.

GHG emissions from the proposed Project are discussed in detail in Section 3 of this application. A facility plot plan with existing and proposed equipment labeled is included as Figure 2-1. A process flow diagram for the proposed unit is included as Figure 2-2 and a process flow diagram that shows integration of the proposed unit with the existing combined cycle unit is included as Figure 2-3.

There are no upstream or downstream impacts that would preclude addition of the proposed unit to the SHEC, because the existing the plant natural gas piping and infrastructure is designed to handle a second GE 7FA combustion turbine and duct burner. The existing steam turbine was designed to achieve full capacity with a second GE 7FA and HRSG, which would improve the heat rate and thermal efficiency of the unit, providing more electricity per unit of natural gas consumed. The existing balance of plant equipment including circulating water, condensate water, cooling water systems and the cooling tower were designed to support an additional 7FA and HRSG.







The existing condenser was constructed to support steam flow from a second HRSG operating in bypass. The plant switchyard is designed to support the electrical production of the additional unit. The plant access road is adequate to support construction and maintain operation of the additional unit. There will be small increases of natural gas fugitives from piping associated with the proposed CTG.

The cooling tower, which uses water from the adjacent City of Austin waste water treatment plant and or potable water or river water, will require additional make-up water. There would also be an increased volume of process water and equipment cooling water usage. There will also be small increases in wastewater due to blow-down from the new HRSG.

2.4 Process Description

The GE 7FA.04 CTG consists of a compressor, burners, turbine and generator on a single shaft as shown in Figure 2-2. Ambient air is introduced to the unit after inlet air filtration and (on high temperature days) evaporative cooling, where an atomized mist of water is used to reduce the air temperature, increasing air density and thus increasing the output of the turbine. Filtered (and cooled) air is compressed in the compressor section prior to combustion with PNG in the combustion zone. Products of combustion from the burner go to the turbine section where they expand to rotate the turbine that drives the compressor and the generator. The exhaust gas exits the turbine at approximately 1100°F and is delivered to the HRSG via ductwork. The HRSG design is a 3-pressure reheat design with high-pressure (HP), intermediate pressure (IP) and low pressure (LP) sections. A duct burner may be used to deliver additional heat to the HP section of the HRSG by combustion of pipeline natural gas using residual oxygen in the flue gas. Heat recovered in the HRSG will be utilized to produce steam.

High pressure steam generated within the HRSG will be used to drive the existing STG and associated electrical generator attached to the same shaft. After expansion in the steam turbine, saturated steam goes to a condenser and is cooled back to water before being returned to the HRSG for reuse. The condenser is cooled via a closed cycle cooling water loop that uses a cooling tower to maintain the circulating water temperature low enough for effective condenser operation. The mechanical induced draft cooling tower uses large fans to draw air into the tower and across the path of the water so that direct contact and heat transfer is made between the hot water and cooler air. Some of the cooling water is lost via evaporation and drift (droplets) and some additional water is lost to blow-down (used to keep solids concentrations from building up) and must be made up via introduction of make-up water to the circulating cooling water. The cooling tower is equipped with mist eliminators to minimize drift and conserve water.

2.4.1 Emission Control Equipment for the Combined Cycle Unit

The emission control technologies proposed for the combustion turbine and duct burner exhaust gases include DLN combustors located within the combustion turbine and an SCR system located within the HRSG to control NO_x emissions. An oxidation catalyst and efficient combustion controls will be used to control emissions of CO and VOC. Emissions of other pollutants are minimized through the proposed use of low-sulfur pipeline natural gas, as well as efficient combustion in the combustion turbine and duct burner.

2.5 Natural Gas Piping

Austin Energy is proposing to utilize PNG as the only fuel for the proposed combustion turbine and duct burner. The natural gas is delivered to the site via an existing natural gas pipeline that serves the site. Gas will be metered and piped to the new combustion turbine and duct burner. The natural gas is assumed to have a HHV of 1,022 Btu/standard cubic foot (scf) and a maximum sulfur content of 0.23 grains per 100 scf. Fugitive emissions from any new gas piping components associated with the new combined cycle unit will include emissions of methane and carbon dioxide, components of the natural gas.

2.6 Electrical Equipment Insulated with Sulfur Hexafluoride (SF₆)

Sulfur hexafluoride (SF₆) is a fluorinated compound with an extremely stable molecular structure. The unique chemical properties of SF₆ make it an efficient electrical insulator. The gas is used for electrical insulation, arc quenching and current interruption in high voltage electrical equipment. The capacity of the generator circuit breaker associated with the proposed unit will be approximately 59 pounds SF₆ is only used in sealed and safe systems which under normal circumstances do not leak gas; however, we account for potential emissions from this equipment in this application to be conservative.

The proposed circuit breaker at the generator output will have a low pressure alarm and a low pressure lockout. The alarm will alert operating personnel of any leakage in the system and the lockout prevents any operation of the breaker due to lack of "quenching and cooling" SF₆ gas.

Section 3 Greenhouse Gas Emission Calculations

3.1 GHG Emissions from Combined Cycle Combustion Turbine

Combustion turbine performance and emissions are affected by ambient temperature and turbine load. Performance, exhaust and emission were developed for the combustion turbine firing natural gas at three combustion turbine steady-state loads (minimum, 75%, and 100%), three different ambient temperatures (0 °F for worst-case winter conditions, 68 °F for average annual conditions, and 112 °F for worst-case summer conditions), and for the effect of evaporative cooling based on GE's Gas Turbine Performance Simulation program. Combined cycle stack parameters and emissions that take into account HRSG duct firing are based on GE Energy's preliminary heat balance estimate for the S207FA.04 combined cycle integrated with the existing unit for design ambient conditions of 68°F and 60% relative humidity at base load with the evaporative cooling on and no duct burner. Emissions estimates and stack parameters for a total of 14 total combustion turbine steady-state operating scenarios were extrapolated from the design heat balance and presented in Appendix B, Table 1.

GHG emissions for the proposed combined cycle combustion turbine (see Table 3-1) are calculated in accordance with the procedures outlined in the Mandatory Greenhouse Gas Reporting Rules. Carbon dioxide (CO₂), methane (CH₄) and nitrous oxide (N₂O) emissions are calculated using the emission factors for natural gas combustion from Table C-1 and Table C-2 of 40 CFR Part 98. The global warming potential factors used to calculate carbon dioxide equivalent (CO₂e) emissions are based on 40 CFR Part 98, Table A-1.

3.2 GHG Emissions from Natural Gas Piping Fugitives and Natural Gas Maintenance and Startup/Shutdown Related Releases

GHG emission calculations for natural gas piping component fugitive emissions (see Table 3-2) are based on emission factors from Table W-1A of the Mandatory Greenhouse Gas Reporting Rules. The concentrations of CH₄ and CO₂ in the natural gas are based on a typical natural gas analysis. The global warming potential factors used to calculate CO₂e emissions are based on Table A-1 of 40 CFR Part 98.

GHG emission calculations for releases of natural gas related to piping maintenance and turbine startup/shutdowns are calculated using the same CH₄ and CO₂ concentrations as natural gas piping fugitives.

Table 3-1
Annual GHG Emissions – Combustion Turbine Combined Cycle Unit

Source	Annual Heat Input¹ (MMBtu/yr)	Pollutant	Emission Factor ² (kg/MMBtu)	GHG Mass Emissions (tons/yr)	Global Warming Potential ³	CO₂e (tons/yr)				
Combustion	22,716,339	CO ₂	53.02	1,327,623.8	1	1,327,624				
turbine plus duct burner		CH ₄	1.0E-03	25.0	25	626				
		N ₂ O	1.0E-04	2.5	298	746				
						GHG	Totals	1,327,651		1,328,996
		CO2e	+10% margin a	1,461,896						

Annual heat input based on 8760 hours per year of operation of the combustion turbine at an average ambient temperature of 680F with evaporative cooling on and a duct burner firing at 681.5 MMBtu/hr for 8760 hours per year.

Example Calculations:

Combustion Turbine Fuel = $8760 \, hr/yr \, x \, 1911.6 \, MMBtu/hr = 16,746,049 \, MMBtu/hr$

Duct Burner Fuel = $8760 \, hr/yr \, x \, 681.5 \, MMBtu/hr = 5,970,290 \, MMBtu/yr$

Total Fuel = 16,746,049 + 5,970,290 = 22,716,339 MMBtu/yr

Carbon Dioxide

 $22,716,339 \ MMBtu/hr \ x \ (53.02 \ kg \ CO2/MMBtu) \ x \ (2.2046 \ lb/kg) \ / \ (2000 \ lb/ton) \ x \ 1 \ CO2e \ / CO2 = 1,327,624 \ tons \ CO2e/yr$

Methane

 $22,716,339 \ MMBtu/hr \ x \ (0.001 \ kg \ CH4/MMBtu) \ x \ (2.2046 \ lb/kg) \ / \ (2000 \ lb/ton) \ x \ 25 \ CO2e \ / CH4 = 626 \ tons \ CO2e/yr$

Nitrous Oxides

 $22,716,339 \, MMBtu/hr \, x \, (0.0001 \, kg \, N20/MMBtu) \, x \, (2.2046 \, lb/kg) \, / \, (2000 \, lb/ton) \, x \, 298 \, CO2e \, /N2O \, = \, 746 \, tons \, CO2e/yr$

² CO₂, CH₄ and N₂O emission factors based on Tables C-1 and C-2 of 40 CFR 98

Global warming potential factors based on Table A-1 of 40 CFR 98

Table 3-2 Annual GHG Emission Calculations – Natural Gas Piping

Source	Component Type	Fluid State	Count	Emission Factor ¹ scf/hr/comp	CO ₂ ² tons/yr	CH₄³ tons/yr	Total tons/yr
Additional	Valves	Gas/Vapor	194	0.121	0.093	4.017	
Natural Gas Fugitives	Flanges		161	0.017	0.011	0.468	
	Relief Valve		35	0.193	0.027	1.156	
GHG Mass Based Emissions						5.642	
Global Warming Potential ⁴						25	
CO ₂ e Emissions						141.04	141.2

- ¹ Emission factors from Table W-1A of 40 CFR Part 98 Mandatory Greenhouse Gas Reporting
- ² CO₂ emissions based on vol% CO₂ in natural gas of 0.79%
- 3 CH₄ emissions based on vol% CH₄ in natural gas of 94.14%
- Global warming potential based on Table A-1 of 40 CFR Part 98 Mandatory Greenhouse Gas Reporting

Example Calculations:

CO₂ Emissions from Valves

$$194 \text{ valves } x \left(0.121 \frac{\text{scf gas}}{\text{hr} - \text{valve}}\right) x \left(0.0079 \frac{\text{scf CO2}}{\text{scf gas}}\right) x \left(\frac{\text{lb} - \text{mole CO2}}{385.5 \text{ scf CO2}}\right) x \left(44 \frac{\text{lb CO2}}{\text{lb} - \text{mole}}\right) x \left(8760 \frac{\text{hr}}{\text{yr}}\right) \\ x \left(\frac{\text{ton}}{2000 \text{ lb}}\right) = 0.093 \frac{\text{ton CO2}}{\text{year}}$$

CH₄ Emissions from Valves

$$194 \text{ valves } x \left(0.121 \frac{\text{scf gas}}{\text{hr} - \text{valve}}\right) x \left(0.9414 \frac{\text{scf CH4}}{\text{scf gas}}\right) x \left(\frac{\text{lb} - \text{mole CH4}}{385.5 \text{ scf CH4}}\right) x \left(16 \frac{\text{lb CH4}}{\text{lb} - \text{mole}}\right) x \left(8760 \frac{\text{hr}}{\text{yr}}\right) \\ x \left(\frac{\text{ton}}{2000 \text{ lb}}\right) = 4.017 \frac{\text{tons CH4}}{\text{year}}$$

3.3 GHG Emissions from Electrical Equipment Insulated with SF₆

SF₆ emissions from the new generator circuit breaker associated with the proposed unit (see Table 3-3) are calculated using a predicted SF₆ annual leak rate of 0.5% by weight per year, the IEC standard for new equipment leakage⁸. The global warming potential factors used to calculate CO₂e emissions are based on Table A-1 of 40 CFR Part 98.

3.4 Total Project GHG Emissions

Table 3-4 summarizes total Project GHG emissions based on the sum of CO₂e emissions for the proposed combined cycle unit, natural gas pipeline fugitives and SF6 emissions from the new generator circuit breaker. Emissions are speciated as CO₂, CH₄, N₂O and SF6 and converted to equivalent CO₂e and summed to calculate total project GHG.

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⁸ International Electrotechnical Commission Standard 62271-1, 2004.

 $\begin{tabular}{ll} Table 3-3\\ GHG Emission Calculations - Electrical Equipment Insulated with SF_6 \end{tabular}$

Estimated Quantity of SF6 in New Equipment	59 pounds		
Annual Leak Rate	0.50% of quantity present		
Annual Emission Rate	0.295 lb/yr		
Annual SF6 Emissions	0.0001475 ton/yr of SF6		
Global Warming Potential Factor for SF6	22,800		
Annual CO2e Emissions	3.36 ton/yr of CO2e		

Example Calculation:

59 lb SF6 x
$$\left(\frac{0.50\%}{\text{yr}}\right)$$
 x $\left(\frac{\text{ton}}{2000 \text{ lb}}\right)$ x 22,800 $\frac{\text{CO2e}}{\text{SF6}} = 3.36 \frac{\text{tons CO2e}}{\text{yr}}$

Table 3-4 Annual GHG Emissions - Total Project

	Annual Potential Emissions, tons/year					
Source	CO ₂	CH₄	N₂O	SF ₆	GHG, CO2e	
Combined Cycle Unit (with 10% margin)	1,460,386	27.5	2.8	0	1,461,896	
Natural Gas Pipeline Fugitives	0.13	5.64	0	0	141.2	
Electrical Equipment Leaks	0	0	0	0.00015	3.36	
Total Project	1,460,386	33.2	2.8	0.0001475	1,462,040	

Section 4 Prevention of Significant Deterioration Applicability

4.1 Applicability

The existing Sand Hill Energy Center is an existing major stationary source as defined under the Clean Air Act. PSD applies to GHG emissions from a proposed modification to an existing major source if the following is true:

■ The emissions increase and the net emissions increase of GHGs from the modification would be equal to or greater than 75,000 tons per year on a CO₂e basis and greater than zero tons per year on a mass basis.

Because the proposed Project's net emissions increase of GHG is greater than 75,000 ton/year of CO₂e (see Table 3-4), PSD review is triggered for GHG emissions. The emissions netting analysis is documented on the TCEQ PSD netting tables (Table 1F and 2F) included in Appendix A.

4.2 Requirements

The PSD regulations state that facilities subject to PSD review must perform an air quality analysis (which can include atmospheric dispersion modeling and preconstruction ambient air quality monitoring), and BACT analysis for those pollutants that exceed the pollutant-specific significant emission rates (SERs) identified in the regulations as well as an additional impacts analysis that examines the impacts of air emissions from the project on visibility, soils and vegetation.

4.2.1 Best Available Control Technology

The proposed Project must utilize BACT controls for pollutants subject to PSD from each piece of new equipment. BACT is defined as the optimum level of control applied to pollutant emissions based upon consideration of energy, economic and environmental factors. In a BACT analysis, the energy, environmental, and economic factors associated with each alternate control technology are evaluated, in addition to the benefit of reduced emissions that the technology would bring. The BACT analysis for the proposed facility is detailed in Section 5.

4.2.2 Air Quality Impact Analysis

An impact analysis is not being provided with this application in accordance with EPA's recommendations:

Because there are no NAAQS or PSD increments for GHGs, the requirements in sections 52.21(k) and 51.166(k) of EPA's regulations to demonstrate that a source does not cause or contribute to a violation of the NAAQS are not applicable to GHGs. Therefore, there is no requirement to conduct dispersion modeling or ambient monitoring for CO2 or GHGs.⁹

An impact analysis for non-GHG emissions is being submitted with the State/ PSD/ Nonattainment application submitted to the TCEQ.

4.2.3 GHG Preconstruction Monitoring

A pre-construction monitoring analysis for GHG is not being provided with this application in accordance with EPA's recommendations:

EPA does not consider it necessary for applicants to gather monitoring data to assess ambient air quality for GHGs under section 52.21(m)(1)(ii), section 51.166(m)(1)(ii), or similar provisions that may be contained in state rules based on EPA's rules. GHGs do not affect "ambient air quality" in the sense that EPA intended when these parts o EPA's rules were initially drafted. Considering the nature of GHG emissions and their global impacts, EPA does not believe it is practical or appropriate to expect permitting authorities to collect monitoring data for the purpose of assessing ambient air impact of GHGs.¹⁰

A pre-construction monitoring analysis for non-GHG emissions is being submitted with the State/PSD/Nonattainment application submitted to the TCEQ.

4.2.4 Additional Impacts Analysis

A PSD additional impacts analysis is not being provided with this application in accordance with EPA's recommendations:

Furthermore, consistent with EPA's statement in the Tailoring Rule, EPA believes it is not necessary for applicants or permitting authorities to assess impacts from GHGs in the context of the additional impacts analysis or Class I area provisions of the PSD regulations for the following policy reasons. Although it is clear that GHG emissions contribute to global warming and other climate changes that result in impacts on the environment, including impacts on Class I areas and soils and vegetation due to the global scope of

⁹ EPA, PSD and Title V Permitting Guidance For Greenhouse Gases at 48-49

¹⁰ Id. at 49

the problem, climate change modeling and evaluations of risks and impacts of GHG emissions is typically conducted for changes in emissions orders of magnitude larger than the emissions from individual projects that might be analyze in PSD permit reviews. Quantifying the exact impacts attributable to a specific GHG source obtaining a permit in specific places and points would not be possible with current climate change modeling. Given these considerations, GHG emissions would serve as the more appropriate and credible proxy for assessing the impact of a given facility. Thus, EPA believes that the most practical way to address the considerations reflected in the Class I area and additional impacts analysis is to focus on reducing GHG emissions to the maximum extent. In light of these analytical challenges compliance with the BACT analysis is the best technique that can be employed a present to satisfy the additional impacts analysis and Class I area requirements of the rule related to GHGs.¹¹

A PSD additional impacts analysis for non-GHG emissions is being submitted with the State/PSD/Nonattainment application submitted to the TCEQ.

4.2.5 Other Required Analyses

Under Section 7 of the Endangered Species Act, Section 106 of the National Historic Preservation Act and Executive Order 12898 [59 FR 7629 (February 16, 1994)], any action authorized or permitted by EPA must ensure that it will not adversely impact federally listed endangered species, historic areas and disproportionately minority or low income populations, respectively. Austin Energy has developed a Biological Assessment and Cultural Resources Survey for the proposed Project. This assessment and all relevant correspondence are included in Appendix D.

¹¹ Id

Section 5 GHG Control Technology Analysis

5.1 Overview

Pre-construction review for new major stationary sources involves an evaluation of BACT. The BACT methodology presented herein is based on USEPA's recommended "top-down," 5-step analysis process to evaluate the available and applicable emission control technologies for the affected pollutants. BACT as defined in 40 CFR Part 52.21(b)(12) is:

"An Emissions limitation (including a visible emission standard) based on the maximum degree of reduction for each pollutant subject to regulation under the Act which would be emitted from any proposed major stationary source or major modification which the Administrator, on a case-by-case basis, taking into account energy, environmental, and economic impacts and other costs, determines is achievable for such source or modification through application of production processes or available methods, systems, and techniques, including fuel cleaning or treatment or innovative fuel combustion techniques for control of such pollutant."

At the head of the list in the top-down analysis are the control technologies and emissions limits that represent the Lowest Achievable Emission Rate (LAER) determinations, which, under NSR/PSD regulations, represent the most effective control alternative and must be considered under the BACT analysis process. BACT cannot be determined to be less stringent than the emission limits established by an applicable New Source Performance Standard (NSPS) for the affected air emission source, but no NSPS limit has been finalized for GHG emissions from utility sources. The methodology uses a 5-step process, which is summarized below.

Step 1: Identify All Control Technologies

The first step in a "top-down" analysis is to identify all available control options, including lower emitting processes, practices, and post-combustion controls.

Step 2: Eliminate Technically Infeasible Options

The second step of the "top-down" analysis is to eliminate the technically infeasible control options from those identified in Step 1, including options that have not been "demonstrated"; or more specifically, a technology that has not been installed and operated successfully on a similar type of unit of comparable size. Technologies that are in development and testing stages are classified as not available and therefore infeasible.

Step 3: Rank Remaining Control Technologies by Control Effectiveness

The third step of the "top-down" analysis is to rank all the remaining (feasible) control alternatives based on their control effectiveness. The following informational databases, clearinghouses, documents, and studies were used to identify recent control technology determinations for similar source categories and emission units:

- USEPA's RACT/BACT/LAER Clearinghouse (RBLC)
- Federal/State/Local new source review permits
- Technical journals, newsletters, and reports
- Information from air quality control technology suppliers
- Engineering design studies for this and similar units

Step 4: Evaluate Most Effective Controls

The additional evaluations of feasible controls consider and compare the energy, environmental, and economic impacts associated with implementing the viable control alternatives. The economic impact analysis is performed to assess the cost to purchase and operate the control technology. The capital cost and operating/annual costs are estimated and a total equivalent annualized cost (\$/year) is divided by the quantity of pollutant removed (tons/year) to calculate the cost effectiveness (\$/ton) of a control technology.

Step 5: Select BACT

The highest ranked control technology from Step 3 that is not eliminated in Step 4 based on unacceptable economic, energy, or environmental impacts, is proposed as BACT for the pollutant and emission unit under review.

5.2 BACT Analysis for Greenhouse Gas (GHG) Emissions

As described in Section 3, the sources of GHG emissions for the proposed Project include the new combined cycle combustion turbine/duct burner, natural gas pipeline fugitives and electrical equipment insulated with SF₆.

5.2.1 BACT Analysis for Combined Cycle Unit

5.2.1.1 Identification of GHG Control Options

A search of the RBLC for "carbon dioxide" did not yield any results for combined cycle combustion turbines; however, Austin Energy is aware of several projects which have recently been permitted with GHG emission limits. These facilities are identified in Table 5-1 below.

Table 5-1
Summary of GHG Permit Emission Limits for Combined Cycle Units

				Combined Cycl Limi	Simple Cycle	
Facility	State	Combustion Turbine Manufacturer / Model	Permit Status	Combined Cycle Output Basis Ib CO2e / MWh	Combined Cycle Heat Rate Btu/kWh (HHV)	Estimated Simple Cycle Heat Rate, Btu/kWh (HHV)*
Russell City	CA	Siemens 501F	Final		7,730	10,385
Pacificorp	UT	Siemens 5000F	Final	950		10,385
LCRA Ferguson	TX	GE 7FA	Final	918	7,720	10,009
Kennecott	UT	GE 7FA	Final		7,642	10,009
Pioneer Valley	MA	Mitsubishi 501G	Final	895		
CPV Valley	NY	Siemens 5000F	Draft	925		10,385
CPV Woodbridge	NJ	GE 7FA	Final	925	7,605	10,009
Cricket Valley	NY	GE 7FA	Final	913**	7,605	10,009
Hess Newark	NJ	GE 7FA	Final	887	7,522	10,009
Calpine Deer Park	TX	Siemens 501F	Final	920	7,730	10,385
Calpine Channel	TX	Siemens 501F	Final	920	7,730	10,385

^{*}Representative base load simple cycle (combustion turbine only) heat rate for model of combustion turbine used / not project-specific data or permit limit

^{**}No output-based (lb CO2e/MWh) limit in permit, but this lb CO2e/MWh value is calculated based on permit limits on heat rate (Btu/kWh) and emission factor (lb/MMBtu):

lb CO2e/MWh = lb CO2e/MMBtu x Btu/kWh x 10³ kWh/MWh x MMBtu/10⁶ BTU

Additional details on these combined cycle facility permits and limits are provided in Appendix C, Table 1.

Based on engineering knowledge and judgment and permit applications submitted to EPA Region 6 for similar facilities, the following potentially applicable GHG control technologies were evaluated:

- Carbon Capture and Storage
- Electrical Generation Efficiency

Carbon Capture and Storage (CCS)

Capture and compression, transport, and geologic storage of the CO₂ is a post-combustion technology that is not considered commercially viable for natural gas combustion sources at this time. However, because EPA Region 6 has requested consideration of CCS by other GHG permit applicants, CCS is evaluated further in this analysis including an evaluation of its cost-effectiveness. CCS systems involve the use of adsorption or absorption processes to remove CO₂ from flue gas, with subsequent desorption to produce a concentrated CO₂ stream. The concentrated CO₂ is then compressed to supercritical temperature and pressure, a state in which CO₂ exists as neither a liquid nor a gas, but instead has physical properties of both liquids and gases. The supercritical CO₂ would then be transported to a location that allows for underground injection into a suitable geological storage reservoir, such as a deep saline aquifer or depleted coal seam, or used in crude oil production for enhanced oil recovery (EOR).

Electrical Generation Efficiency

Other than capture and sequestration of GHG emitted by combustion, the only known option for reducing GHG emissions from natural gas combined cycle units is through maximization of the energy released during the combustion process and then through the maximization of the use or capture of that energy. To minimize GHG emissions, it is desirable to use less fuel to generate a given amount of electrical energy. There are several factors that may affect the amount of GHG produced per MW-hour of energy produced. These include low carbon fuels and the thermodynamic and mechanical efficiency of the combustion unit.

5.2.1.2 Elimination of Technically Infeasible Alternatives

All options identified above are considered "technically" feasible for the purposes of this BACT analysis.

5.2.1.3 Ranking of Remaining Technologies Based on Effectiveness

Both CCS and electrical generation efficiency are considered "technically" feasible and are ranked in order of desirability below:

- 1. Carbon Capture and Storage (CCS)
- 2. Electrical Generation Efficiency

5.2.1.4 Evaluation of Control Technologies

The energy, environmental, and economic feasibility of implementing the different control technologies as BACT for GHG emissions from the proposed Project's gas turbine/HRSG train are evaluated below for the two technically feasible control technologies:

- 1. Carbon Capture and Storage (CCS)
- 2. Electrical Generation Efficiency

Carbon Capture and Storage (CCS)

Each component of CCS technology (i.e., capture and compression, transport, and storage) is discussed separately. These include:

- 1. Carbon Capture and Compression
- 2. CO₂ Transport
- 3. CO₂ Storage

Carbon Capture and Compression

Although amine absorption technology for CO₂ capture has been applied to processes in the petroleum refining and natural gas processing industries and to exhausts from gas-fired industrial boilers, it is more difficult to apply to power plant gas turbine exhausts, which have considerably larger flow volumes and considerably lower CO₂ concentrations (partial pressures). The Obama Administration's Interagency Task Force on Carbon Capture and Storage confirms this in its recently completed report on the current status of development of CCS systems:

"Current technologies could be used to capture CO₂ from new and existing fossil energy power plants; however, they are not ready for widespread implementation primarily because they have not been demonstrated at the scale necessary to establish confidence for power plant application. Since the CO₂ capture capacities used in current industrial processes are generally much smaller than the capacity

required for the purposes of GHG emissions mitigation at a typical power plant, there is considerable uncertainty associated with capacities at volumes necessary for commercial deployment."¹²

In its current CCS research program plans, the Department of Energy's National Energy Technology Laboratory (DOE-NETL) confirms that commercial CO₂ capture technology for large-scale power plants is not yet available and suggests that it may not be available until at least 2020:

"The overall objective of the Carbon Sequestration Program is to develop and advance CCS technologies that will be ready for widespread commercial deployment by 2020." To accomplish widespread deployment, four program goals have been established:

- 1. Develop technologies that can separate, capture, transport, and store CO2 using either direct or indirect systems that result in a less than 10 percent increase in the cost of energy by 2015;
- 2. Develop technologies that will support industries' ability to predict CO2 storage capacity in geologic formations to within ±30 percent by 2015;
- 3. Develop technologies to demonstrate that 99 percent of injected CO2 remains in the injection zones by 2015;
- 4. Complete Best Practices Manuals (BPMs) for site selection, characterization, site operations, and closure practices by 2020. Only by accomplishing these goals will CCS technologies be ready for safe, effective commercial deployment both domestically and abroad beginning in 2020 and through the next several decades."13

The partial pressure for CO₂ in a natural gas combined cycle flue gas is much lower than the partial pressure of CO₂ in a coal-fired boiler flue gas: the volume of CO₂ in a natural gas combined cycle stack is typically 3-5% compared to 11-12% in a coal fired boiler stack flue gas.

Most carbon capture research and trials to date feature the amine absorption process for coal fired power plants, because the flue gas has a much higher CO₂ partial pressure in a coal boiler. Other disadvantages of the amine process that have not been fully addressed are the following:

- 1. The high concentration of oxygen in the natural gas turbine stack adversely affects solvents such as amine.
- 2. The amine processes also tend also to be prohibitively expensive, for one because of the cooling that is required to remove sulfur and nitrogen compounds that may potentially poison the solvent.

¹² Report of the Interagency Task Force on Carbon Capture and Storage at 50 (Aug. 2010).

¹³ DOE-NETL, Carbon Sequestration Program: Technical Program Plan, at 10 (Feb. 2011).

Significant research conducted by Mitsubishi Heavy Industries (MHI) has demonstrated both the advantage of inhibited amine technology and the significantly greater cost and difficulty for removal of CO2 from gas turbines as compared to coal boilers. Further support for the conclusion that commercial availability of CO2 capture technology for large-scale power plant projects will not occur for several more years was found by reviewing information published by Alstom, a major developer of commercial CO2 capture technology using post-combustion amine absorption, post-combustion chilled ammonia absorption and oxy-combustion. Alstom states on its web site that its CO2 capture technology will become commercially available in 2015.¹⁴ However, it should be noted that in committing to this timeframe, the company does not indicate whether such technology will be able to handle the volume of CO2 emissions generated by a project the size of this proposed project.

Conservation of water resources is another important challenge associated with CO₂ capture. A modern natural gas fired combined cycle facility requires four to five million gallons of water per day for condenser cooling and boiler make-up service. This amount will vary based on ambient temperature and humidity as well as the level of duct firing in the HRSG. Adding CO₂ separation facilities and compression equipment significantly increases the cooling water requirements of a generating station. Studies indicate that the water consumption of a natural fired combined cycle facility with CCS may have an increased water consumption of more than 60%.¹⁵

CO₂ Transport

Even if it is assumed that CO₂ capture and compression could feasibly be achieved for the proposed Project, the high-volume CO₂ stream generated would need to be transported to a facility capable of storing it. Potential geologic storage sites in Texas, Louisiana, and Mississippi to which CO₂ could be transported if a pipeline was constructed are illustrated in Figure 5-1. The potential length of such a CO₂ transport pipeline cannot be determined because no sites can be identified that are suitable for large-scale, long-term CO₂ storage. However, Denbury Resources (Denbury) operates a CO₂ pipeline in southeast Texas, called the "Green Pipeline". Debury will purchase high purity CO₂ from companies and will use the CO₂ for enhanced oil recovery (EOR). The nearest location for delivery to the Green Pipeline is in the Hastings oil field southeast of Houston. The distance to the Hastings oil field is 135 miles. In order to get CO₂ from SHEC to the Green Pipeline Austin energy would have to construct a 135 mile pipeline.

Alstom, Alstom's Carbon Capture Technology Commercially "Ready to Go" by 2015, Nov. 30, 2010, http://www.alstom.com/australia/news-and-events/pr/ccs2015/

¹⁵ Report of the Interagency Task Force on Carbon Capture and Storage at 50 (Aug. 2010).

Frio Test Site SECARB Phase II & III Cranfield Pipelines for naturally occuring CO₂ **ARKANSAS** NEW **ALABAMA** MISSISSIPPI **MEXICO** Natural CO₂ source Ft. Worth-Dallas Travis Peak Hosston TEXAS **FLORIDA** Austin Houston **SHEC Site SACROC Southwest** Eocene, Partnership NM San Antonio Tech CO₂-EOR candidate reservoirs 100 200 miles Existing CO₂ pipelines 300 kilometers Additional oil-production area with CO2-EOR production and potential Major oil plays Oligocene QAd4485ax

Figure 5-1: Potential CO₂ Storage Areas and Existing CO₂ Pipelines

Source: University of Texas at Austin Bureau of Economic Ecology, Gulf Coast Carbon Center

CO₂ Storage

Even if it is assumed that CO₂ capture and compression could feasibly be achieved for the proposed project and that the CO₂ could be transported economically, the feasibility of CCS technology would still depend on the availability of a suitable sequestration site. The suitability of potential storage sites is a function of volumetric capacity of their geologic formations, CO₂ trapping mechanisms within formations (including dissolution in brine, reactions with minerals to form solid carbonates, and/or adsorption in porous rock), and potential environmental impacts resulting from injection of CO₂ into the formations. Potential environmental impacts resulting from CO₂ injection that still require assessment are significant unknown risks before CCS technology can be considered feasible include:

- Uncertainty concerning the significance of dissolution of CO₂ into brine,
- Risks of brine displacement resulting from large-scale CO₂ injection, including a pressure leakage risk for brine seeping into underground drinking water sources and/or surface water,
- Risks to fresh water as a result of leakage of CO₂, including the possibility for damage to the biosphere, underground drinking water sources, and/or surface water, ¹⁶
- Potential effects on wildlife, and
- Risk of metals leaking from underground formations as a result of the injection of acid gases.

Potentially suitable storage sites, including EOR sites and saline formations, exist in Texas, Louisiana and Mississippi. As was discussed above, the nearest location where the infrastructure exists in the Hastings oil field 135 miles away from SHEC.

Based on the reasons provided above, the City of Austin believes that CCS technology should be eliminated from further consideration as a potential feasible control technology for purposes of this BACT analysis. However, to answer possible questions that the public or the EPA may have concerning the relative costs of implementing hypothetical CCS systems, the City of Austin has estimated such costs.

Cost and Energy Impacts of Carbon Capture and Storage

The estimated costs associated with implementation of a carbon capture system for the proposed Project are shown in Table 5-2 below. Capital cost components include equipment

¹⁶ Susan Hovorka, University of Texas at Austin, Bureau of Economic Geology, Gulf Coast Carbon Center, *New Developments: Solved and Unsolved Questions Regarding Geologic Sequestration of CO2 as a Greenhouse Gas Reduction Method* (GCCC Digital Publication #08-13) at slide 4 (Apr. 2008), available at: http://www.beg.utexas.edu/gccc/forum/codexdownloadpdf.php?ID=100

(complete with initial chemical and catalyst loadings), materials, labor (direct and indirect), engineering and construction management and contingencies. Operating cost components include operating and maintenance labor, consumables, fuel, waste disposal and co-product or by-product credit.

The capital cost of a CO₂ capture system for SHEC would be approximately \$170 million¹⁷. Based on an estimated project capital cost for the addition of a CTG and HRSG of \$195 million, the addition of the CO₂ capture portion of CCS control scheme would be expected to add 87% to the project's overall capital cost. The capital cost for the pipeline that would be required to transport the CO₂ to a suitable storage site is estimated to be another \$238 million.

In addition to the high construction and operating costs associated with CCS, the carbon capture equipment requires a substantial amount of energy to operate, thereby reducing the net electrical output of the plant. Operation of carbon capture equipment at a typical natural gas fired combined cycle plant is estimated to reduce the net energy efficiency of the plant from approximately 50% to approximately 42.9%.¹⁸

The capital cost to construct a suitable 135-mile pipeline to the nearest site with any potential for geological storage of CO₂ is calculated to be greater than \$238 million (Appendix C, Table 2) per pipeline cost equations developed by DOE-NETL.¹⁹

¹⁷ Report of the Interagency Task Force on Carbon Capture and Storage (Aug. 2010).

¹⁸ US Department of Energy, National Energy Technology Laboratory, "Costs and Performance Baseline For Fossil Energy Plants, Volume 1 - Bituminous Coal and Natural Gas to Energy", Revision 2, November 2010

¹⁹ Quality Guidelines for Energy System Studies: Estimating Carbon Dioxide Transport and Storage Costs, National Energy Technology Laboratory, US Department of Energy, DOE/NETL-2010/1447 (March, 2010)

Table 5-2 CCS Cost Analysis for Combined Cycle Unit

	Capital Cost ¹ (\$)	Annual Cost ¹ (\$/ton)	CO ₂ Removed ³ (ton/yr)	Total Annualized Cost ⁴ (\$/year)
CCS System Component	(1)	(2)	(3)	(2) * (3)
CO ₂ Capture & Compression Facilities	\$170,000,000	\$103	1,314,348	\$135,928,752
CO ₂ Transport Facilities	\$238,205,778	\$18.00²	1,314,348	\$23,651,945
CO₂ Storage Facilities	NA ⁵	\$0.51	1,314,348	\$667,720
Total CCS System Cost	>\$408,000,000	\$122	1,314,348	\$160,248,417
Potential Revenue from Sale of CO ₂ for EOR ⁶		(\$23)	1,314,348	(\$30,229,994)
Total CCS System Cost with Sale of CO ₂ for EOR		\$99	1,314,348	\$130,018,423

- CO₂ capture/compression and storage costs are from Report of the Interagency Task Force on Carbon Capture (August, 2010). Cost includes initial investment, O&M, and cost of fuel.
- CO₂ transport cost is calculated per pipeline cost equations from Quality Guidelines for Energy System Studies: Estimating Carbon Dioxide Transport and Storage Costs, National Energy Technology Laboratory, US Department of Energy, DOE/NETL-2010/1447 (March, 2010). See Appendix Table 2
- 3. Tons of CO₂ removed assumed 90% capture of all CO₂ emissions from the combustion turbine and duct burner.
- 4. Annualized costs based on 7% interest rate and 20 year equipment life
- 5. No capital cost estimate provided for CO2 storage in Report of the Interagency Task Force on Carbon Capture (August, 2010)
- US price from sale of CO₂ for Enhanced Oil Recovery (EOR) is \$10-40/tonne, from http://www.globalccsinstitute.com/publications/global-status-ccs-2012/online/48431

Example Calculations (Total Annualized Cost):

Capture & Comp:
$$\left(\frac{\$103}{ton}\right) \times \left(\frac{1,314,348 ton CO2}{yr}\right) = \frac{\$135,928,752}{yr}$$

CO2 Transport:
$$\left(\frac{\$18.00}{ton}\right) \times \left(\frac{1,314,348 ton CO2}{yr}\right) = \frac{\$23,651,945}{yr}$$

CO2 Storage:
$$\left(\frac{\$0.51}{\text{ton}}\right) \times \left(\frac{1,314,348 \text{ ton CO2}}{\text{yr}}\right) = \frac{\$667,720}{\text{yr}}$$

$$\text{Total CCS System Cost: } \left(\frac{\$135,928,752}{\text{yr}} \right) + \left(\frac{\$23,561,945}{\text{yr}} \right) + \left(\frac{\$667,720}{\text{yr}} \right) = \ \frac{\$144,597,870}{\text{yr}}$$

Based on the same guidance, the annual pipeline O&M cost would be more than \$1,165,320/year, and the total annual cost including annualized capital cost would be about \$23.7 million per year. A cost is also associated with the monitoring and maintenance of the CO₂ storage facility. The total equivalent annual cost for all aspects of the site-specific CCS option total about \$160 million/year. Assuming the system would capture 90% of the combined cycle unit CO₂, the cost effectiveness is calculated at \$122/ton CO₂. Assuming that all of the CO₂ captured could be sold for use in Enhanced Oil Recovery (EOR), a \$23/ton credit has been applied in the cost analysis, reducing the net cost of CCS to \$99/ton CO₂.

Electrical Generation Efficiency

The following energy efficiency practices are associated with and incorporated into the proposed combined cycle unit design:

Use of Low Carbon Fuel

The first aspect to evaluate with regard to an energy efficient process is the source of fuel. 40 CFR part 98 provides emission factors for GHG from the combustion of various fuels. Natural gas is listed as the third cleanest fuel with respect to CO₂ emissions, the third cleanest fuel with respect to CH₄ emissions and the cleanest fuel with respect to N₂O emissions. The two cleaner fuels with respect to CO₂ emissions (coke oven gas and biogas) are not feasible sources of fuel for the project based on the need to match the technology choice (natural gas turbine) to the existing combined cycle unit. Therefore, with regard to fuels that can be utilized by the project, natural gas produces the lowest GHG emissions profile.

Turbine Design/Selection

In a combined cycle configuration, a HRSG is used to recover what would otherwise be waste heat lost to the atmosphere in the hot turbine exhaust. Use of heat recovery from the turbine exhaust to produce steam to power a steam turbine which generates additional electric power is the single most effective means of increasing the efficiency of combustion turbines used for electric power generation. The overall thermal efficiency for the proposed project is increased from about 39% for the simple cycle combustion turbine (no heat recovery) unit to about 59% for the unit in combined cycle configuration including electricity generated in the STG. In applications where process heat is needed, the steam produced in the HRSG can also be used to provide heat to plant processes in addition to or instead of being used to produce additional electricity. This "cogeneration" technology is not applicable to electric power generation unless there is a co-located steam host or other means of using additional recoverable waste heat.

The existing combined cycle unit at SHEC is operating in a 1×1 configuration with a GE 7FA gas turbine with a 164 MW (nominal at ISO) generator, a HRSG, and a GE D-11 steam turbine with a 189 MW generator. Not all of this STG capacity can be utilized with only the existing CTG/HRSG, even with the duct burner firing at maximum design capacity. Ultimate expansion of the unit to a 2×1 configuration was part of the original design and the existing steam turbine and cooling tower were sized for a second 7FA and associated HRSG.

In evaluating different turbines, SHEC decided that a second 7FA provided benefits from an operations and maintenance perspective that other turbines models did not. SHEC plant staff is experienced in operating and maintaining the 7FA and the addition of a second 7FA would minimize changes to the control system, operating procedures and training. A second 7FA would also simplify maintenance by leveraging staff experience as well as reduce the quantity and cost of spare parts which would be required to maintain two different gas turbines. Estimated cost savings associated with selection of a second GE 7FA unit versus a different turbine model are:

- Operation training: \$300,000
- Control system integration: \$3,000,000 (exclusive of engineering and design changes and commissioning complications)
- Spare combustion parts: \$3,000,000 to \$5,000,000
- Long Term Service Agreement (LTSA): \$3,000,000 to \$4,000,000 (incremental)

A newer version of the GE 7FA turbine is now available that allows for a higher firing temperature and therefore improvements in output and heat rate (i.e., greater thermal efficiency). The output of the existing GE 7FA.03 CTG is 161.5 MW at full load at the site ambient average temperature of 68°F and the GE 7FA.04 output is 200 MW at ISO conditions. Although there would be advantages for SHEC to have identical GE 7FA.03 turbines at the facility, the improved performance of the 7FA.04 will make the overall efficiency of the combined cycle unit with the shared STG better and thereby reduce the GHG emissions in terms of CO2e/MWh of electricity generated.

Heat rate (Btu/kWh) values are provided for the combustion turbine alone (simple cycle) and in combination with the HRSG/STG output (combined cycle) for each of the 14 operating modes modeled for the proposed new unit in the spreadsheet of performance and emissions data provided in Appendix B, Table B-1, and summarized in Table 5-3 below. Example calculations for heat rates are provided in the footnotes to Table B-1.

Table 5-3 Estimated Performance and Emissions for GE 7FA.04 Combustion Turbine in Simple Cycle and Combined Cycle Mode

pad Condition	% CTG Base Load	BASE	BASE	75%	50%	BASE	BASE	BASE	75%	53%	BASE	BASE	BASE	75%	69%
mbient Temperature	deg F	0	0	0	0	68	68	68	68	68	112	112	112	112	112
vap. Cooler Status		Off	Off	Off	Off	On	On	Off	Off	Off	On	On	Off	Off	Off
vap. Cooler Effectiveness	%	0	0	0	0	85	85	0	0	0	85	85	0	0	0
uct Burner Status		On	Off	Off	Off	On	Off	Off	Off	Off	On	Off	Off	Off	Off
mbient Relative Humidity	%	60	60	60	60	60	60	60	60	60	60	60	60	60	60
let Loss	in H2O	4	4	4	4	4	4	4	4	4	4	4	4	4	4
khaust Pressure Loss	in H2O	16.1	16.1	10.7	7.3	14.4	14.4	14	9.5	7	11.5	11.5	10.7	8	7.4
TG Output (Simple Cycle)	kW	196,178	196,178	147,133	98,089	179,025	179,025	173,910	130,433	92,172	152,206	152,206	142,802	107,101	98,533
TG Output (estimated)*	kW	168,683	102,365	86,067	74,672	162,271	95,953	94,620	80,141	71,053	153,844	87,526	85,269	75,042	72,240
C (CTG + STG) Combined Cycle Output	kW	364,861	298,543	233,200	172,761	341,296	274,978	268,530	210,574	163,225	306,050	239,732	228,071	182,143	170,77
TG Simple Cycle Heat Rate (HHV)	BTU/kWh, HHV	9,744	9,744	10,462	12,470	9,964	9,964	10,020	10,823	12,611	10,454	10,454	10,639	11,825	12,19
TG Simple Cycle Heat Rate, HHV (+ margin**)	BTU/kWh, HHV	10,718	10,718	11,508	13,717	10,961	10,961	11,022	11,906	13,872	11,500	11,500	11,703	13,008	13,41
ombined Cycle (CTG + STG) Heat Rate (HHV)	BTU/kWh, HHV	7,104	6,403	6,601	7,080	7,221	6,488	6,489	6,704	7,121	7,422	6,637	6,662	6,953	7,038
ombined Cycle (CTG + STG) Heat Rate (+ margin**)	BTU/kWh, HHV	7,815	7,044	7,261	7,788	7,943	7,136	7,138	7,375	7,833	8,165	7,301	7,328	7,648	7,742
TG Heat Consumption (HHV)	MMBTU/hr	1,911.6	1,911.6	1,539.4	1,223.2	1,784.0	1,784.0	1,742.4	1,411.7	1,162.4	1,591.1	1,591.1	1,519.4	1,266.4	1,201.
D2	lb/hr	303,110	223,447	179,934	142,976	288,187	208,524	203,666	165,011	135,864	265,647	185,984	177,602	148,028	140,48
02	lb/MMBTU, HHV	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9
20	lb/hr	0.6	0.4	0.3	0.3	0.5	0.4	0.4	0.3	0.3	0.5	0.4	0.3	0.3	0.3
20	lb/MMBTU, HHV	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.0002
1 4	lb/hr	5.7	4.2	3.4	2.7	5.4	3.9	3.8	3.1	2.6	5.0	3.5	3.3	2.8	2.6
H4	lb/MMBTU, HHV	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.002
HG, CO2e	lb/hr	303,408	223,666	180,110	143,116	288,470	208,728	203,866	165,173	135,997	265,907	186,166	177,776	148,173	140,62
HG, CO2e	lb/MMBTU, HHV	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0
HG, CO2e (Simple Cycle)	ton/MWh	0.773	0.570	0.612	0.730	0.806	0.583	0.586	0.633	0.738	0.874	0.612	0.622	0.692	0.714
HG, CO2e (Simple Cycle + margin**)	ton/MWh	0.851	0.627	0.673	0.802	0.886	0.641	0.645	0.696	0.812	0.961	0.673	0.685	0.761	0.785
HG, CO2e (Combined Cycle)	ton/MWh	0.416	0.375	0.386	0.414	0.423	0.380	0.380	0.392	0.417	0.434	0.388	0.390	0.407	0.412
HG, CO2e (Combined Cycle + margin**)	ton/MWh	0.457	0.412	0.425	0.456	0.465	0.417	0.418	0.431	0.458	0.478	0.427	0.429	0.447	0.453
STG output (kW) estimated by scaling from design case perating, but maximum duct burner firing rate used for a				o of exhaus	t energy + d	uct burner h	eat input to	design case	e exhaust er	ergy; actua	STG outpu	t limited to 1	48 MW with	only 1 CTC	è

Simple cycle gross heat rates range from 10,718 to 11,703 Btu/kWh (HHV) at full load without evaporative cooling. Combined cycle heat rates are estimated to range from 7,044 to 7,328 Btu/kWh (HHV) for these same operating conditions²⁰. Heat rates are higher (efficiency lower) at higher ambient temperatures and lower turbine loads. The use of evaporative cooling improves the heat rate by lowering the compressor inlet temperature. Use of the duct burners to increase steam production in the HRSG and STG output increases the overall combined cycle heat rate.

It is worth noting that the outputs and heat rates identified above are based on the specific turbine design and performance for this project at the SHEC site. Thus it is likely that these values may differ somewhat from those published in the manufacturer's literature or website summarizing performance for a variety of combustion turbine manufacturers and models. Factors that influence the output and heat rate from project to project include ambient temperature, humidity and site elevation; fuel type/composition and exhaust back pressure (due to the HRSG, catalysts and the length of the flue gas path – or stack height) will also have a significant impact on performance. The combined cycle heat rate of the proposed unit compares favorably to the heat rate limits included in recent permits for other GE 7FA units, as well as combined cycle units with other GE turbine models and those of other manufacturers, which are summarized in Table 5-1. Most of these permits have minimum combined cycle heat rate limits in the range of 7,522 to 7,730 Btu/kWh, HHV.

GHG emission limits for units comparable to the proposed SH8 unit that have output-based lb/MWh and or heat rate (Btu/kWh) basis limits are extracted from the comprehensive list in Appendix C and summarized in Table 5-1 above.

5.2.1.5 Determination of BACT for Combined Cycle Unit

The City of Austin (dba Austin Energy) proposes as BACT for this project, the following energy efficiency processes, practices, and designs for the proposed combined cycle combustion turbine:

- Use of Combined Cycle Power Generation Technology
- Use of natural gas as the exclusive fuel
- Efficient turbine design

²⁰ SH8 combined cycle heat rates are based on STG outputs extrapolated from the design (full load at 68 degrees F) case used to model the performance for a 2-on-1 configuration with the existing SH5 unit; in some cases with duct firing the STG output is greater than it can actually be in order to evaluate maximum duct burner firing rates.

The proposed GE 7FA.04 combustion turbine is the most efficient unit available that is suitable for incorporation into the existing combined cycle unit as described above, with a gross heat rate at base load ranging from 9,744 to 10,639 Btu/kWh, HHV, at base load without evaporative cooling, depending on ambient conditions (refer to Table 5-3). For comparison, the existing SH5 unit combustion turbine, as GE 7FA.03 model, has a heat rate ranging from 10,004 to 11,055 Btu/kWh, HHV, across the same range of conditions, so the newer model is about 3 to 4% more efficient, translating to 3 to 4% lower GHG emissions. Other manufacturers have made similar improvements: a chief competitor to the GE 7FA, the Siemens 5000F (Westinghouse 501F) turbine; has improved simple cycle efficiency from approximately by approximately 4% over the last 20 or so years. In combined cycle mode, with the benefit of the HRSG and additional output from the STG, the gross heat rate of the proposed GE 7FA.04 unit is expected range to from 7,044 to 7,328 Btu/kWh, HHV without evaporative cooling.

Because the new combustion turbine will be added to an existing combined cycle unit and its HRSG will provide steam to a steam turbine generator that is shared with the existing turbine, it is not possible to determine a combined cycle heat rate standard and output-based GHG limit that is separate from the performance of the existing SH5 turbine and HRSG performance. The SH5 unit was permitted prior to GHG BACT requirements and as such has no GHG emission limit or heat rate limit in its permit conditions. Therefore, the City of Austin is proposing GHG BACT limits based on the new combustion turbine alone (simple cycle basis) without duct burner firing:

- Simple cycle output-based GHG limit of <u>0.81 ton CO₂e/MWh</u> combustion turbine gross output basis; equivalent to 0.458 ton CO₂e/MWh (or 916 lb/MWh) gross output basis for the combined cycle unit that shares the STG with the existing SH5 combustion turbine, with unfired HRSG
- Simple cycle combustion turbine gross output heat rate of 13,872 Btu/kWh (HHV); equivalent to 7,833 BTU/kWh (HHV) gross heat rate for the combined cycle unit that shares the STG with the existing SH5 combustion turbine, at 68°F with unfired HRSG

The proposed output-based GHG emission limits are equivalent to the most recent GHG permit limits for similar combined cycle units as summarized in Table 5-1 (approximately 0.466 ton CO2e/MWh or 932 lb/MWh, combined cycle basis) and the proposed heat rate limit is comparable to the most recent limits for similar units.

As with other recent GHG permits for combined cycle units, compliance with these performance standards is determined for operation at 100% load at ISO conditions, without evaporative cooling or duct burner firing. This provides a standard basis for comparison of the GHG limits that are permitted for different projects. In order to demonstrate that the proposed simple cycle GHG limit corresponds to the combined cycle performance indicated above, the

City of Austin proposes to perform the initial compliance test with only the new combined cycle unit operating (i.e., with SH5 off line). In order to regulate GHG emissions from all operating conditions, including those using evaporative cooling and with duct burner firing, the City of Austin is also proposing a GHG BACT limit on total annual GHG emissions for the combined cycle combustion turbine and duct burner:

■ Annual emission cap of 1,461,896 tons CO₂e per year, for the addition of the second combustion turbine, HRSG and duct burner

The proposed GHG BACT limits for the proposed new unit are summarized in Table 5-4 below.

This approach of a combination of output based CO₂e limits and heat rate limits at standard conditions and an annual emission cap to cover other operating conditions and emissions has been used in other recent GHG BACT permit limits for combined cycle units.

Each of the proposed limits is calculated to include a 10% margin to account for measurement error, equipment and site variations, and degradation over time.

Compliance with the output based emission limits will be based on a 30-day rolling average. Compliance with the annual tons/year limit will be based on a rolling 365-day total. In each case, SHEC proposes to calculate GHG emissions based on continuous monitoring of the fuel flow to the combustion turbine and duct burner using calibrated fuel flow meters, with heat input calculated based on weighted average monthly heating values provided by the pipeline natural gas supplier, and emissions of CO₂e calculated based on emission factors and GWPs from 40 CFR 98 (greenhouse gas monitoring rule). Gross output (kW) will be continuously measured and recorded at the combustion turbine generator.

5.2.2 BACT Analysis for Natural Gas Fugitives

Small amounts of methane may occur from leaking natural gas piping components (process fugitives) associated with the proposed Project. The methane emissions from process fugitives have been conservatively estimated to be 141.2 tons per year (see Table 3-2) as CO₂e. This is a negligible (0.010%) contribution to the total GHG emissions from the project. However, for completeness, they are addressed in this BACT analysis.

5.2.2.1 Identification of GHG Control Options

The only identified control technology for process fugitive emissions of CO₂e is a leak detection and repair (LDAR) program. LDAR programs vary in stringency/degree of control as needed for control of VOC emissions; however due to the negligible amount of GHG emissions from fugitives, LDAR programs would not be considered for control of GHG emissions alone.

Table 5-4
Proposed Greenhouse Gas BACT Limits for SH8 (Simple Cycle Output Basis)

Form of Limit	Limit	Averaging Period	Basis
Output-Based GHG Limit	0.81 tons CO₂e/MWh ¹	30-day rolling average	Simple cycle combustion turbine only gross output basis at full load
Heat Rate Limit	13,872 Btu/kWh (HHV) ²	30-day rolling average	Simple cycle combustion turbine only gross output basis at full load
Annual GHG Emission Cap	1,461,896 tons CO₂e/year	365-day rolling average	Includes all stack emissions from combustion turbine, duct burner, start-ups, shutdowns, malfunctions and effects of different operating conditions including evaporative inlet air cooling

- Simple cycle (combustion turbine only) output based GHG limit is equivalent to 0.458 ton CO₂e/MWh (or 916 lb/MWh) for the combined cycle unit at full load at 68 degrees F
- Simple cycle (combustion turbine only) heat rate limit is equivalent to 7,833 BTU/kWh (HHV) gross heat rate for the combined cycle unit at full load at 68 degrees F

5.2.2.2 Elimination of Technically Infeasible Alternatives

LDAR is considered technically feasible for the proposed Project.

5.2.2.3 Ranking of Remaining Technologies Based on Effectiveness

The use of a LDAR program with a portable gas analyzer meeting the requirements of 40 CFR 60, Appendix A, Method 21, can be effective for identifying leaking methane. Quarterly instrument monitoring with a leak definition of 10,000 part per million by volume (ppmv) (TCEQ 28M LDAR Program) is generally assigned a control efficiency of 75% for valves, relief valves, sampling connections, and compressors and 30% for flanges. Quarterly instrument monitoring with a leak definition of 500 ppmv (TCEQ 28VHP LDAR Program) is generally assigned a control efficiency of 97% for valves, relief valves, and sampling connections, 85% for compressors, and 30% for flanges. The EPA has allowed the use of an optical gas imaging instrument as an alternative work practice for a Method 21 portable analyzer to monitor equipment for leaks in 40 CFR 60.18(g). For components containing inorganic or odorous compounds, periodic audio/visual/olfactory (AVO) walk-through inspections provide predicted control efficiencies of 97% control for valves, flanges, relief valves, and sampling connections, and 95% for compressors. Can be effective for identifying leaking methane. Quarterly instrument (Ppmv)

5.2.2.4 Evaluation of Control Technologies

A cost effectiveness analysis for a basic LDAR program to control process fugitive CH₄ emissions is presented in Appendix C, Table 3. The results of this analysis show that the least stringent LDAR program (TCEQ's 28M program) would cost approximately \$260 per ton of CO₂e removed, which is not considered to be cost effective for GHG control.

5.2.2.5 Determination of BACT for Natural Gas Fugitives

Due to the negligible amount of GHG emissions from process fugitives, the only available control, implementation of an LDAR program, is clearly not cost effective and would result in no significant reduction of overall project GHG emissions regardless of cost. Based on these considerations, BACT is determined to be normal plant maintenance practices as needed for safety and reliability purposes.

²¹ Air Permit Technical Guidance for Chemical Sources: Equipment Leak Fugitives, TCEQ, Oct. 2000

²² Id

²³ Id

5.2.3 BACT Analysis for SF₆ Insulated Electrical Equipment

5.2.3.1 Identification of GHG Control Options

State-of-the-art enclosed-pressure SF₆ technology with leak detection is the primary technology used to limit fugitive emissions. In comparison to older SF₆ circuit breakers, modern breakers are designed as a totally enclosed-pressure system with far lower potential for SF₆ emissions. In addition, the effectiveness of leak-tight closed systems can be enhanced by equipping them with a density alarm that provides a warning when 10% of the SF₆ (by weight) has escaped. The use of an alarm identifies potential leak problems before the bulk of the SF₆ has escaped, so that it can be addressed proactively to prevent further release of the gas.

A second alternative considered in this analysis is to substitute another, non-GHG substance for SF₆ as the dielectric material in the breakers. Potential alternatives to SF₆ were addressed in the National Institute of Standards and Technology (NTIS) Technical Note 1425, *Gases for Electrical Insulation and Arc Interruption: Possible Present and Future Alternatives to Pure SF*₆, ²⁴

5.2.3.2 Elimination of Technically Infeasible Alternatives

According to the NTIS Technical Note 1425, SF₆ is a superior dielectric gas for nearly all high voltage applications.²⁵ It is easy to use, exhibits exceptional insulation and arc-interruption properties, and has proven its performance by many years of use and investigation. It is clearly superior in performance to the air and oil insulated equipment used prior to the development of SF₆-insulated equipment. The report concluded that although "…various gas mixtures show considerable promise for use in new equipment, particularly if the equipment is designed specifically for use with a gas mixture… it is clear that a significant amount of research must be performed for any new gas or gas mixture to be used in electrical equipment." Therefore there are currently no technically feasible options besides use of SF₆.

5.2.3.3 Ranking of Remaining Technologies Based on Effectiveness

The use of state-of-the-art SF₆ technology with leak detection to limit fugitive emissions is the highest ranked control technology that is technically feasible for this application.

²⁴ Christophorous, L.G., J.K. Olthoff, and D.S. Green, *Gases for Electrical Insulation and Arc Interruption: Possible Present and Future Alternatives to Pure SF6*, NIST Technical Note 1425, Nov.1997.

²⁵ Id at 28-29

5.2.3.4 Evaluation of Control Technologies

Energy, environmental, or economic impacts were not addressed in this analysis because the use of alternative, non-greenhouse-gas substances to SF_6 as the dielectric material in the breakers is not technically feasible. A cost-effectiveness evaluation was not preformed given there is only one feasible option.

5.2.3.5 Determination of BACT for GHG for SF₆ Insulated Electrical Equipment

The City of Austin proposes to use circuit breakers with totally enclosed insulation systems equipped with a low pressure alarm and low pressure lockout. The lockout will prevent operation of the breaker if insufficient SF₆ remains in the system.

Appendix A TCEQ Applicable Forms



Important Note: The agency requires that a Core Data Form be submitted on all incoming applications unless a Regulated Entity and Customer Reference Number have been issued and no core data information has changed. For more information regarding the Core Data Form, call (512) 239-5175 or go to www.tceq.texas.gov/permitting/central_registry/guidance.html.

I. Applicant Information					
A. Company or Other Legal Nan	ne: City of Austin dba	Austin Energy			
Texas Secretary of State Charter/Registration Number (if applicable):					
B. Company Official Contact Na	B. Company Official Contact Name: Mr. Ravi Joseph				
Title: Consulting Engineer					
Mailing Address: 721 Barton Springs R	Road				
City: Austin	State: Texas		ZIP Cod	le: 78704	
Telephone No.: 512-322-6284	Fax No.: 512-322-62	80	E-mail	Address: ravi.joseph@austinenergy.com	
C. Technical Contact Name: Mr.	Ravi Joseph				
Title: Consulting Engineer					
Company Name: Austin Energy					
Mailing Address: 721 Barton Springs Roa	ad				
City: Austin	State: Texas		ZIP Code: 78704		
Telephone No.: 512-322-6284	Fax No.: 512-322-6	280	E-mail Address: ravi.joseph@		
D. Site Name: Sand Hill Energy Ce	enter				
E. Area Name/Type of Facility:	Electric Generation Fa	acility		☐ Permanent ☐ Portable	
F. Principal Company Product of	or Business: Electric	Power Production, T	ransmissi	on and Distribution	
Principal Standard Industrial Classif	fication Code (SIC):	: 4911			
Principal North American Industry (Classification System	m (NAICS):			
G. Projected Start of Construction	on Date: 2nd Quarter	r of 2015			
Projected Start of Operation Date: 2r	nd Quarter of 2017				
H. Facility and Site Location Inf in writing.):	ormation (If no str	eet address, provid	le clear d	lriving directions to the site	
Street Address: 1102 Fallwell Lane					
City/Town: Del Valle	County: Travis		ZIP Cod	le: 78617	
Latitude (nearest second): 30°12′28″		Longitude (neares	t second	l): 97°36' 53"	





I.	Applicant Information (continued)		
I.	Account Identification Number (leave blank if new site or facility):		
J.	Core Data Form.		
	Core Data Form (Form 10400) attached? If No, provide customer refere gulated entity number (complete K and L).	nce number	☐ YES ☐ NO
K.	Customer Reference Number (CN):		
L.	Regulated Entity Number (RN):		
II.	General Information		
A.	Is confidential information submitted with this application? If Yes, mar confidential page confidential in large red letters at the bottom of each		☐ YES ☐ NO
В.	Is this application in response to an investigation, notice of violation, o action? If Yes, attach a copy of any correspondence from the agency and RN in section I.L. above.		☐ YES ☐ NO
C.	Number of New Jobs:		
D.	Provide the name of the State Senator and State Representative and dissite:	strict numbers 1	or this facility
State S	enator:	District No.:	
State F	Representative:	District No.:	
III.	Type of Permit Action Requested		
A.	Mark the appropriate box indicating what type of action is requested.		
☐ Init	ial Amendment Revision (30 TAC 116.116(e) Change of	of Location	Relocation
В.	Permit Number (if existing):		
C.	Permit Type: Mark the appropriate box indicating what type of permit (check all that apply, skip for change of location)	is requested.	
☐ Con	nstruction 🔲 Flexible 🔲 Multiple Plant 🔲 Nonattainment 🔲 Pl	lant-Wide Appl	icability Limit
☐ Pre	vention of Significant Deterioration Hazardous Air Pollutar	nt Major Source	
☐ Oth	er:		
D.	Is a permit renewal application being submitted in conjunction with the amendment in accordance with 30 TAC 116.315(c).	is	☐ YES ☐ NO





III.	. Type of Permit Action Requested	(conti	nued)				
E.	Is this application for a change of locat If Yes, complete III.E.1 - III.E.4.0	Is this application for a change of location of previously permitted facilities? If Yes, complete III.E.1 - III.E.4.0					
1.	Current Location of Facility (If no street address, provide clear driving directions to the site in writing.):						
Stre	eet Address:						
City	y: County:			ZIP Code:			
2.	Proposed Location of Facility (If no street	addres	s, provide clear drivi	ing directions to the	e site in writing.):		
Stre	eet Address:						
City	y: County:			ZIP Code:			
3.	Will the proposed facility, site, and plot pl the permit special conditions? If "NO", at			al requirements of	☐ YES ☐ NO		
4.	Is the site where the facility is moving con or HAPs?	sidered	a major source of co	riteria pollutants	☐ YES ☐ NO		
F.	Consolidation into this Permit: List an consolidated into this permit including						
List	t:						
G.	Are you permitting planned maintenar attach information on any changes to e in VII and VIII.				☐ YES ☐ NO		
Н.	Federal Operating Permit Requiremen (30 TAC Chapter 122 Applicability) Is this facility located at a site required operating permit? If Yes, list all associa attach pages as needed).	l to obta		☐ YES ☐ NO ☐ 7	To be determined		
Ass	sociated Permit No (s.):						
1.	Identify the requirements of 30 TAC Chap	oter 122	that will be triggere	d if this application	is approved.		
☐ I	FOP Significant Revision	Minor	☐ Application for	an FOP Revision			
	Operational Flexibility/Off-Permit Notifica	ition	☐ Streamlined Re	vision for GOP			
	To be Determined		None				





III.	Type of Permit Action	Requested (continued)	
	· ·	Requirements (30 TAC Chapter 122 Applicability) (continu	ued)
2. Ide	1 0	issued and/or FOP application(s) submitted/pending for t	
GOF	P Issued	☐ GOP application/revision application submitted or unc	ler APD review
SOP	'Issued	SOP application/revision application submitted or und	er APD review
IV.	Public Notice Applicab	ility	
A.	Is this a new permit applic	ation or a change of location application?	☐ YES ☐ NO
B.	Is this application for a con	ncrete batch plant? If Yes, complete V.C.1 – V.C.2.	☐ YES ☐ NO
	Is this an application for a FCAA 112(g) permit, or exc	major modification of a PSD, nonattainment, ceedance of a PAL permit?	☐ YES ☐ NO
		D or major modification of a PSD located within n affected state or Class I Area?	☐ YES ☐ NO
If Yes, l	list the affected state(s) and	l/or Class I Area(s).	
List:			
Е.	Is this a state permit amen	dment application? If Yes, complete IV.E.1. – IV.E.3.	
1. Is t	there any change in charac	ter of emissions in this application?	☐ YES ☐ NO
2. Is t	there a new air contaminar	nt in this application?	☐ YES ☐ NO
	the facilities handle, load, gumes, or vegetables fibers	unload, dry, manufacture, or process grain, seed, (agricultural facilities)?	☐ YES ☐ NO
		ion increases associated with the application d attach additional sheets as needed):	
Volatile	e Organic Compounds (VO	C):	
Sulfur l	Dioxide (SO2):		
Carbon	Monoxide (CO):		
Nitroge	en Oxides (NOx):		
Particu	late Matter (PM):		
PM 10 1	microns or less (PM10):		
PM 2.5	microns or less (PM2.5):		
Lead (F	Pb):		
Hazard	lous Air Pollutants (HAPs)	:	
Other s	speciated air contaminants	not listed above:	





V.	Public Notice Informatio	n (complete if applicable)				
Α.	Public Notice Contact Name:					
Title:						
Mailir	ng Address:					
City:		State:	ZIP Code:			
B.	Name of the Public Place:					
Physic	cal Address (No P.O. Boxes):					
City:		County:	ZIP Code:			
The processory		zation to place the application for pul	olic viewing and	☐ YES ☐ NO		
The p	ublic place has internet access	available for the public.		☐ YES ☐ NO		
C.	Concrete Batch Plants, PSD,	and Nonattainment Permits				
	ounty Judge Information (For acility site.	Concrete Batch Plants and PSD and/o	or Nonattainment	Permits) for this		
The H	onorable:					
Mailir	ng Address:					
City:	y: State: ZIP Code:					
	s the facility located in a munic nunicipality? <i>(For Concrete I</i>	ipality or an extraterritorial jurisdictic Batch Plants)	on of a	☐ YES ☐ NO		
Presid	ling Officers Name(s):					
Title:						
Mailir	ng Address:					
City:		State:	ZIP Code:			
		ess of the chief executive and Indian C e location where the facility is or will b		nd identify the		
Chief	Executive:					
Mailir	ng Address:					
City:		State:	ZIP Code:			
Name	of the Indian Governing Body	:				
Mailir	ng Address:					
City:		State:	ZIP Code:			





V.	Public Notice Information (complete if applicable) (continued)				
C.	Concrete Batch Plants, PSD, and Nonattainment Permits				
3.	Provide the name, mailing address of the chief executive and Indian Governing Body; and identify the Federal Land Manager(s) for the location where the facility is or will be located. <i>(continued)</i>				
Naı	me of the Federal Land Manager(s):				
D.	Bilingual Notice				
Is a	bilingual program required by the Texas Education Code in the School District?	☐ YES ☐ NO			
	the children who attend either the elementary school or the middle school closest to a facility eligible to be enrolled in a bilingual program provided by the district?	☐ YES ☐ NO			
If Y	Yes, list which languages are required by the bilingual program?				
VI.	Small Business Classification (Required)				
A.	Does this company (including parent companies and subsidiary companies) have fewer than 100 employees or less than \$6 million in annual gross receipts?	☐ YES ☐ NO			
В.	Is the site a major stationary source for federal air quality permitting?	☐ YES ☐ NO			
C.	Are the site emissions of any regulated air pollutant greater than or equal to 50 tpy?	☐ YES ☐ NO			
D.	Are the site emissions of all regulated air pollutants combined less than 75 tpy?	☐ YES ☐ NO			
VI	I. Technical Information				
Α.	The following information must be submitted with your Form PI-1 (this is just a checklist to make sure you have included everything)				
1.	☐ Current Area Map				
2.	☐ Plot Plan				
3.	☐ Existing Authorizations				
4.	☐ Process Flow Diagram				
5.	☐ Process Description				
6.	☐ Maximum Emissions Data and Calculations				
7.	☐ Air Permit Application Tables				
a.	☐ Table 1(a) (Form 10153) entitled, Emission Point Summary				
b.	☐ Table 2 (Form 10155) entitled, Material Balance				
c.	☐ Other equipment, process or control device tables				
В.	Are any schools located within 3,000 feet of this facility?	☐ YES ☐ NO			





VII.	Technical Information			
C.	Maximum Operating Schedule:			
Hour(s	Day(s):	Week(s):	Year(s):	
Seasor	al Operation? If Yes, please describe in the space	e provide below.		☐ YES ☐ NO
D.	Have the planned MSS emissions been previous inventory?	sly submitted as part of an e	missions	☐ YES ☐ NO
	e a list of each planned MSS facility or related ac acluded in the emissions inventories. Attach pag		ars the M	SS activities have
				I— —
Е.	Does this application involve any air contamina required?	ints for which a disaster revi	ew is	☐ YES ☐ NO
F.	Does this application include a pollutant of con(APWL)?	cern on the Air Pollutant Wa	tch List	☐ YES ☐ NO
VIII.	State Regulatory Requirements Applicants must demonstrate complianc a permit or amendment. The application mapplicability or non applicability; identify statinclude compliance demonstrations.	ust contain detailed attachr	nents ado	dressing
A.	Will the emissions from the proposed facility procomply with all rules and regulations of the TCI		are, and	☐ YES ☐ NO
В.	Will emissions of significant air contaminants f	rom the facility be measured	l?	☐ YES ☐ NO
C.	Is the Best Available Control Technology (BACT	Γ) demonstration attached?		☐ YES ☐ NO
D.	Will the proposed facilities achieve the perform application as demonstrated through recordkee other applicable methods?			☐ YES ☐ NO
IX.	Federal Regulatory Requirements Applicants must demonstrate compliance obtain a permit or amendment. The application applicability or non applicability; identify federate; and include compliance demonstrations.	cation must contain detailed	l attachm	ents addressing
Α.	Does Title 40 Code of Federal Regulations Part Performance Standard (NSPS) apply to a facility		Source	☐ YES ☐ NO
В.	Does 40 CFR Part 61, National Emissions Stand (NESHAP) apply to a facility in this application		itants	☐ YES ☐ NO





IX.	Federal Regulatory Requirements Applicants must demonstrate compliance with all applicable federal regulations to obtain a permit or amendment. The application must contain detailed attachments addressing applicability or non applicability; identify federal regulation subparts; show how requirements are met; and include compliance demonstrations.					
C.	Does 40 CFR Part 63, Maximum Achievable Control Technolog apply to a facility in this application?	ard	☐ YES ☐ NO			
D.	Do nonattainment permitting requirements apply to this applic	ation?		☐ YES ☐ NO		
Е.	Do prevention of significant deterioration permitting requirements apply to this application?					
F.	Do Hazardous Air Pollutant Major Source [FCAA 112(g)] requir application?	this	☐ YES ☐ NO			
G.	Is a Plant-wide Applicability Limit permit being requested?					
X.	Professional Engineer (P.E.) Seal					
Is the estimated capital cost of the project greater than \$2 million dollars?				☐ YES ☐ NO		
If Yes,	submit the application under the seal of a Texas licensed P.E.					
XI.	Permit Fee Information					
Check,	Money Order, Transaction Number ,ePay Voucher Number:	Fee Amount: \$				
Paid online?				☐ YES ☐ NO		
Compa	ny name on check:					
	Is a copy of the check or money order attached to the original submittal of this application?					
	s a Table 30 (Form 10196) entitled, Estimated Capital Cost and Fee Verification, ttached?					



XII. Delinquent Fees and Penalties

This form will not be processed until all delinquent fees and/or penalties owed to the TCEQ or the Office of the Attorney General on behalf of the TCEQ is paid in accordance with the Delinquent Fee and Penalty Protocol. For more information regarding Delinquent Fees and Penalties, go to the TCEQ Web site at: www.tceq.texas.gov/agency/delin/index.html.

XIII. Signature

The signature below confirms that I have knowledge of the facts included in this application and that these facts are true and correct to the best of my knowledge and belief. I further state that to the best of my knowledge and belief, the project for which application is made will not in any way violate any provision of the Texas Water Code (TWC), Chapter 7, Texas Clean Air Act (TCAA), as amended, or any of the air quality rules and regulations of the Texas Commission on Environmental Quality or any local governmental ordinance or resolution enacted pursuant to the TCAA I further state that I understand my signature indicates that this application meets all applicable nonattainment, prevention of significant deterioration, or major source of hazardous air pollutant permitting requirements. The signature further signifies awareness that intentionally or knowingly making or causing to be made false material statements or representations in the application is a criminal offense subject to criminal penalties.

Name:	
Signature:	Original Signature Required
Date:	

PRINT FORM

RESET FORM



TEXAS COMMISSION ON ENVIRONMENTAL QUALITY

Table 1(a) Emission Point Summary

Date: August 2013	Permit No.: 48106 and PSD-TX1012-M1	Regulated Entity No.: RN100215052
Area Name: City of Austin Sand Hill Energ	y Center (SHEC)	Customer Reference No.: CN600135198

Review of applications and issuance of permits will be expedited by supplying all necessary information requested on this Table.

			AIR CONTAMINANT DATA							
1. Emission P	oint		2. Component or Air Contaminant Name	3. Air Contaminant En	nission Rate					
(A) EPN	(B) FIN	(C) Name		(A) Pound Per Hour	(B) TPY					
SH8	SH8	GE 7FA.04 w/HRSG Combined	CO ₂ e	333,766 lb/hr	1,461,896 TPY					
CC MS FUG	CC MS FUG	Combined Cycle Natural Gas Meter	CO ₂ e	32.2 lb/hr	141 TPY					
SF6 FUG	SF6 FUG	SF6 Fugitives	CO ₂ e	0.77	3 tpy					

EPN = Emission Point Number FIN = Facility Identification Number



TEXAS COMMISSION ON ENVIRONMENTAL QUALITY

Table 1(a) Emission Point Summary

Date: August 2013	Permit No.: 48106 and PSD-TX1012-M1	Regulated Entity No.: RN100215052
Area Name: City of Austin Sand Hill Energ	sy Center (SHEC)	Customer Reference No.: CN600135198

Review of applications and issuance of permits will be expedited by supplying all necessary information requested on this Table.

	IR CONTAMINANT DATA								HARGE PARA					
1. Emis	ssion Poin	nt		TM Coordina mission Point		5. Build			7. Stack Exit	Sour Data	ce	8. Fugitiv	⁄es	
(A) EPN	(B) FIN	(C) NAME	Zone	East (Meters)	North (Meters)	Heigh (Ft.)	t	Above Ground (Ft.)	(A) Diameter (Ft.)	(B) Velocity (FPS)	(C) Temperature (°F)		(B) Width (Ft.)	(C) Axis Degrees
SH8	SH8	GE 7FA.04 w/HRSG	14	633,590	3,342,923			157.5	18.05	40.15	200			
CC MS FUG	CC MS FUG	Combined Cycle Natural Gas Meter	14	633,514	3,342,925			3.0				30	20	40
SF6 FUG	SF6 FUG	SF6 Fugitives	14	633,549	3,342,875			3.0				10	10	100

EPN = Emission Point Number FIN = Facility Identification Number

TCEQ - 10153 (Revised 04/08) Table 1(a)
This form is for use by sources subject to air quality permit requirements and may be revised periodically. (APDG 5178 v5)

Appendix B Emissions Calculations and Documentation

Appendix B, Table 1 Estimated Performance and Emissions for GE 7FA.04 Combustion Turbine in Simple Cycle and Combined Cycle Mode

		MODE NUMBER	1	2	3	4	5	6	7	8	9	10	11	12	13	14
	pad Condition	% CTG Base Load	BASE	BASE	75%	50%	BASE	BASE	BASE	75%	53%	BASE	BASE	BASE	75%	69%
	mbient Temperature	deg F	0	0	0	0	68	68	68	68	68	112	112	112	112	112
~	vap. Cooler Status		Off	Off	Off	Off	On	On	Off	Off	Off	On	On	Off	Off	Off
	vap. Cooler Effectiveness	%	0	0	0	0	85	85	0	0	0	85	85	0	0	0
ш	uct Burner Status		On	Off	Off	Off	On	Off	Off	Off	Off	On	Off	Off	Off	Off
	mbient Relative Humidity	%	60	60	60	60	60	60	60	60	60	60	60	60	60	60
~	let Loss	in H2O	4	4	4	4	4	4	4	4	4	4	4	4	4	4
_	xhaust Pressure Loss	in H2O	16.1	16.1	10.7	7.3	14.4	14.4	14	9.5	7	11.5	11.5	10.7	8	7.4
_	uel Type		Cust Gas													
_,	uel LHV	BTU/lb	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460	20,460
Α,	uel Temperature	deg F	365	365	365	365	365	365	365	365	365	365	365	365	365	365
_)	TG Output (Simple Cycle)	kW	196,178	196,178	147,133	98,089	179,025	179,025	173,910	130,433	92,172	152,206	152,206	142,802	107,101	98,533
_	TG Output (estimated)*	kW	168,683	102,365	86,067	74,672	162,271	95,953	94,620	80,141	71,053	153,844	87,526	85,269	75,042	72,240
	C (CTG + STG) Combined Cycle Output	kW	364,861	298,543	233,200	172,761	341,296	274,978	268,530	210,574	163,225	306,050	239,732	228,071	182,143	170,773
-	TG Simple Cycle Heat Rate (LHV)	BTU/kWh, LHV	8,792	8,792	9,440	11,252	8,991	8,991	9,041	9,766	11,379	9,433	9,433	9,600	10,670	11,007
	TG Simple Cycle Heat Rate (HHV)	BTU/kWh, HHV	9,744	9,744	10,462	12,470	9,964	9,964	10,020	10,823	12,611	10,454	10,454	10,639	11,825	12,199
	TG Simple Cycle Thermal Efficiency	%	35.0%	35.0%	32.6%	27.4%	34.3%	34.3%	34.1%	31.5%	27.1%	32.6%	32.6%	32.1%	28.9%	28.0%
	TG Simple Cycle Heat Rate, LHV (+ margin**)	BTU/kWh, LHV	9,671	9,671	10,384	12,377	9,890	9,890	9,945	10,743	12,517	10,376	10,376	10,560	11,737	12,108
ш	TG Simple Cycle Heat Rate, HHV (+ margin**)	BTU/kWh, HHV	10,718	10,718	11,508	13,717	10,961	10,961	11,022	11,906	13,872	11,500	11,500	11,703	13,008	13,419
	ombined Cycle (CTG + STG) Heat Rate (LHV)	BTU/kWh	6,410	5,778	5,956	6,389	6,515	5,854	5,855	6,049	6,426	6,697	5,989	6,011	6,274	6,351
	ombined Cycle (CTG + STG) Heat Rate (HHV)	BTU/kWh, HHV	7,104	6,403	6,601	7,080	7,221	6,488	6,489	6,704	7,121	7,422	6,637	6,662	6,953	7,038
	ombined Cycle (CTG + STG) Thermal Efficiency	%	48.0%	53.3%	51.7%	48.2%	47.3%	52.6%	52.6%	50.9%	47.9%	46.0%	51.4%	51.2%	49.1%	48.5%
=1	ombined Cycle (CTG + STG) Heat Rate (+ margin**)	BTU/kWh, LHV	7,051	6,355	6,552	7,027	7,167	6,439	6,440	6,654	7,068	7,367	6,588	6,612	6,901	6,986
	ombined Cycle (CTG + STG) Heat Rate (+ margin**)	BTU/kWh, HHV	7,815	7,044	7,261	7,788	7,943	7,136	7,138	7,375	7,833	8,165	7,301	7,328	7,648	7,742
_	TG Heat Cons. (LHV)	MMBTU/hr	1,724.9	1,724.9	1,389.0	1,103.7	1,609.7	1,609.7	1,572.2	1,273.8	1,048.8	1,435.7	1,435.7	1,371.0	1,142.7	1,084.5
_	TG Heat Consumption (HHV)	MMBTU/hr	1,911.6	1,911.6	1,539.4	1,223.2	1,784.0	1,784.0	1,742.4	1,411.7	1,162.4	1,591.1	1,591.1	1,519.4	1,266.4	1,201.9
_)	TG Exhaust Flow	x10^3 lb/hr	3,733	3,733	3,019	2,466	3,493	3,493	3,435	2,804	2,389	3,086	3,086	2,979	2,557	2,464
~	TG Exhaust Temperature	deg F	1,076	1,076	1,116	1,182	1,130	1,130	1,134	1,172	1,215	1,177	1,177	1,188	1,215	1,215
	rhaust Molecular Weight	lb/lbmol	28.48	28.48	28.49	28.5	28.32	28.32	28.35	28.35	28.37	27.87	27.87	27.91	27.93	27.93

Estimated Performance and Emissions for GE 7FA.04 Combustion Turbine in Simple Cycle and Combined Cycle Mode

OMBUSTION TURBINE EMISSIONS															
Ох	ppmvd @ 15% O2	9	9	9	9	9	9	9	9	9	9	9	9	9	9
Ox AS NO2	lb/hr	62	62	50	40	58	58	57	46	38	52	52	50	41	39
D	ppmvd	9	9	9	11	9	9	9	9	11	9	9	9	9	10
þ	lb/hr	31	31	25	24	28	28	28	23	25	24	24	23	20	23
HC	ppmvw	7	7	7	7	7	7	7	7	7	7	7	7	7	7
HC HC	lb/hr	15	15	12	10	14	14	14	11	9	12	12	12	10	10
DC DC	ppmvw	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4	1.4
DC DC	lb/hr	3.0	3.0	2.4	2.0	2.8	2.8	2.8	2.2	1.8	2.4	2.4	2.4	2.0	2.0
M/PM-10/PM-2.5 (filterable)	lb/hr	9	9	9	9	9	9	9	9	9	9	9	9	9	9
M/PM-10/PM-2.5 (f+c)	lb/hr	12	12	12	12	12	12	12	12	12	12	12	12	12	12
)2	lb/MMBTU, HHV	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642	0.000642
02	lb/hr	1.23	1.23	0.99	0.79	1.15	1.15	1.12	0.91	0.75	1.02	1.02	0.98	0.81	0.77
D2 (per 40 CFR 98 Table C-1 EFs)	lb/MMBTU, HHV	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9
D2 (per 40 CFR 98 Table C-1 EFs)	lb/hr	223,447	223,447	179,934	142,976	208,524	208,524	203,666	165,011	135,864	185,984	185,984	177,602	148,028	140,488
2O (per 40 CFR 98 Table C-2 EFs)	lb/MMBTU, HHV	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022
2O (per 40 CFR 98 Table C-2 EFs)	lb/hr	0.42	0.42	0.34	0.27	0.39	0.39	0.38	0.31	0.26	0.35	0.35	0.33	0.28	0.26
H4 (per 40 CFR 98 Table C-2 EFs)	lb/MMBTU, HHV	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022
H4 (per 40 CFR 98 Table C-2 EFs)	lb/hr	4.21	4.21	3.39	2.70	3.93	3.93	3.84	3.11	2.56	3.51	3.51	3.35	2.79	2.65
HG, CO2e (per 40 CFR 98 EFs)	lb/hr	223,666	223,666	180,110	143,116	208,728	208,728	203,866	165,173	135,997	186,166	186,166	177,776	148,173	140,626
HG, CO2e (per 40 CFR 98 EFs)	lb/MMBTU, HHV	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0
OMBUSTION TURBINE EXHAUST ANALYSIS % VOL.															
gon	% by volume	0.90	0.90	0.89	0.90	0.89	0.89	0.88	0.88	0.89	0.85	0.85	0.84	0.84	0.84
itrogen	% by volume	75.08	75.08	75.09	75.17	73.91	73.91	74.12	74.14	74.24	70.67	70.67	71.00	71.08	71.12
xygen	% by volume	12.50	12.50	12.54	12.76	12.25	12.25	12.35	12.42	12.69	11.43	11.43	11.59	11.83	11.94
arbon Dioxide	% by volume	3.91	3.91	3.90	3.79	3.88	3.88	3.86	3.83	3.70	3.85	3.85	3.82	3.71	3.66
rater	% by volume	7.61	7.61	7.58	7.38	9.08	9.08	8.79	8.73	8.49	13.20	13.20	12.75	12.54	12.44
DTAL	,	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0	100.0

Estimated Performance and Emissions for GE 7FA.04 Combustion Turbine in Simple Cycle and Combined Cycle Mode

	UCT BURNER EMISSIONS															
	uct Burner Firing Rate*	MMBTU/hr, LHV	614.0				614.0					614.0				
4	uct Burner Firing Rate*	MMBTU/hr, HHV	681.5				681.5					681.5				1
	Ox	lb/MMBTU, HHV	0.10				0.10					0.10				1
Ч	Ox	lb/hr	68.15				68.15					68.15				
П	D	lb/MMBTU, HHV	0.10				0.10					0.10				
	D	lb/hr	68.15				68.15					68.15				
_	OC (per EPA AP-42 Table 1.4-2)	lb/MMBTU, HHV	0.0054				0.0054					0.0054				
4	DC	lb/hr	3.67				3.67					3.67				
ĺ	M/PM-10/PM-2.5 (F+C) (per EPA AP-42 Table 1.4-2)	lb/MMBTU, HHV	0.0074				0.0074					0.0074				i
	M/PM-10/PM-2.5 (F+C)	lb/hr	5.07				5.07					5.07				i
$\boldsymbol{\tau}$	02	lb/MMBTU, HHV	0.00064				0.00064					0.00064				i
J	02	lb/hr	0.44				0.44					0.44				
	D2 (per 40 CFR 98 Table C-1 EF)	lb/MMBTU, HHV	116.9				116.9					116.9				<u>. </u>
	D2 (per 40 CFR 98 Table C-1 EF)	lb/hr	79,663				79,663					79,663				i
\sim	2O (per 40 CFR 98 Table C-1 EF)	lb/MMBTU, HHV	0.0002				0.0002					0.0002				i
	2O (per 40 CFR 98 Table C-1 EF)	lb/hr	0.15				0.15					0.15				
	H4 (per 40 CFR 98 Table C-1 EF)	lb/MMBTU, HHV	0.0022				0.0022					0.0022				
	H4 (per 40 CFR 98 Table C-1 EF)	lb/hr	1.50				1.50					1.50				
П	HG, CO2e (per 40 CFR 98 EF & GWP)	lb/MMBTU, HHV	117.0				117.0					117.0				
	HG, CO2e (per 40 CFR 98 EF & GWP)	lb/hr	79,741				79,741					79,741				
	RSG EXHAUST COMPOSITION (VOLUME %)					1	ı						1			
	gon	% by volume	0.89%	0.90%	0.89%	0.90%	0.88%	0.89%	0.88%	0.88%	0.89%	0.84%	0.85%	0.84%	0.84%	0.84%
	trogen	% by volume	74.09%	75.08%	75.09%	75.17%	72.87%	73.90%	74.12%	74.14%	74.23%	69.57%	70.67%	71.00%	71.08%	71.12%
-	kygen	% by volume	9.65%	12.50%	12.54%	12.76%	9.22%	12.25%	12.35%	12.42%	12.69%	8.08%	11.43%	11.59%	11.83%	11.94%
7.5	arbon Dioxide	% by volume	5.22%	3.91%	3.90%	3.79%	5.27%	3.88%	3.86%	3.83%	3.70%	5.40%	3.85%	3.82%	3.71%	3.66%
J	ater	% by volume	10.15%	7.61%	7.58%	7.38%	11.75%	9.08%	8.79%	8.73%	8.49%	16.11%	13.20%	12.75%	12.54%	12.44%
~	DTAL		100%	100%	100%	100%	100%	100%	100%	100%	100%	100%	100%	100%	100%	100%
•																
	RSG FLUE GAS MOLECULAR WT												I I			
4	ue Gas	lb/lb-mole	28.33	28.49	28.49	28.50	28.16	28.33	28.35	28.36	28.37	27.69	27.87	27.92	27.93	27.94
	RSG STACK EXHAUST PARAMETERS	6	427.0	427.0	427.0	407.0	407.0	407.0	427.0	407.0	407.0	427.0	427.0	407.0	427.0	127.0
1	ack Height AGL	ft	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0	127.0
	ack Diameter	ft	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0
-	ack Exhaust Temperature	Degrees F	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0	200.0
_	ack Exhaust Flow Rate	ACFM	1,066,827	1,052,528	850,915	694,806	1,004,821	990,522	972,947	794,219	676,263	903,448	889,149	857,090	735,149	708,411
	ack Exit Velocity	ft/s	62.7	61.9	50.0	40.8	59.1	58.2	57.2	46.7	39.8	53.1	52.3	50.4	43.2	41.6

Appendix B, Table 1

Estimated Performance and Emissions for GE 7FA.04 Combustion Turbine in Simple Cycle and Combined Cycle Mode

NCONTROLLED & CONTROLLED EMISSIONS (TURBINE + DUCT BURNER)

TOTAL TOTAL PROPERTY OF THE PR	DOCT DOMINENT														
Ox Uncontrolled	lb/hr	130.2	62.0	50.0	40.0	126.2	58.0	57.0	46.0	38.0	120.2	52.0	50.0	41.0	39.0
Ox Uncontrolled	ppmvd@15%O2	13.6	9.7	9.7	8.3	12.7	8.1	8.1	8.1	8.2	12.0	7.7	7.8	7.7	7.8
Ox Controlled	lb/hr	19.2	12.7	10.4	9.6	19.9	14.4	14.0	11.3	9.3	20.1	13.5	12.8	10.6	10.1
Ox Controlled	ppmvd@15%O2	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0
Uncontrolled	lb/hr	99.2	31.0	25.0	24.0	96.2	28.0	28.0	23.0	25.0	92.2	24.0	23.0	20.0	23.0
Uncontrolled	ppmvd@15%O2	17.0	8.0	7.9	8.2	15.9	6.4	6.6	6.7	8.8	15.1	5.9	5.9	6.2	7.5
D Controlled	lb/hr	11.7	7.8	6.3	5.8	12.1	8.8	8.5	6.9	5.7	12.2	8.2	7.8	6.5	6.1
D Controlled	ppmvd@15%O2	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0	2.0
pc	lb/hr	6.7	3.0	2.4	2.0	6.5	2.8	2.8	2.2	1.8	6.1	2.4	2.4	2.0	2.0
pc	ppmvd@15%O2	2.0	1.4	1.3	1.2	1.9	1.1	1.1	1.1	1.1	1.7	1.0	1.1	1.1	1.1
M/PM-10/PM-2.5 (F+C)	lb/hr	17.1	12.0	12.0	12.0	17.1	12.0	12.0	12.0	12.0	17.1	12.0	12.0	12.0	12.0
M/PM-10/PM-2.5 (F+C)	lb/MMBTU, HHV	0.0066	0.0063	0.0078	0.0098	0.0069	0.0067	0.0069	0.0085	0.0103	0.0075	0.0075	0.0079	0.0095	0.0100
M-10 incl. Ammonia Salts	lb/hr	18.4	13.0	12.8	12.6	18.4	12.9	12.9	12.7	12.6	18.3	12.8	12.8	12.7	12.6
M-10 incl. Ammonia Salts	lb/MMBTU, HHV	0.0071	0.0068	0.0083	0.0103	0.0075	0.0073	0.0074	0.0090	0.0109	0.0080	0.0081	0.0084	0.0100	0.0105
02	lb/hr	1.67	1.23	0.99	0.79	1.58	1.15	1.12	0.91	0.75	1.46	1.02	0.98	0.81	0.77
D2	lb/MMBTU, HHV	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064	0.00064
2SO4	lb/hr	1.02	0.75	0.61	0.48	0.97	0.70	0.69	0.56	0.46	0.89	0.63	0.60	0.50	0.47
2SO4	lb/MMBTU, HHV	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039	0.00039
D2	lb/hr	303,110	223,447	179,934	142,976	288,187	208,524	203,666	165,011	135,864	265,647	185,984	177,602	148,028	140,488
02	lb/MMBTU, HHV	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9	116.9
20	lb/hr	0.6	0.4	0.3	0.3	0.5	0.4	0.4	0.3	0.3	0.5	0.4	0.3	0.3	0.3
20	lb/MMBTU, HHV	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022	0.00022
H4	lb/hr	5.7	4.2	3.4	2.7	5.4	3.9	3.8	3.1	2.6	5.0	3.5	3.3	2.8	2.6
1 4	lb/MMBTU, HHV	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022	0.0022
HG, CO2e	lb/hr	303,408	223,666	180,110	143,116	288,470	208,728	203,866	165,173	135,997	265,907	186,166	177,776	148,173	140,626
HG, CO2e	lb/MMBTU, HHV	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0	117.0
HG, CO2e (Simple Cycle)	ton/MWh	0.773	0.570	0.612	0.730	0.806	0.583	0.586	0.633	0.738	0.874	0.612	0.622	0.692	0.714
HG, CO2e (Simple Cycle + margin**)	ton/MWh	0.851	0.627	0.673	0.802	0.886	0.641	0.645	0.696	0.812	0.961	0.673	0.685	0.761	0.785
HG, CO2e (Combined Cycle)	ton/MWh	0.416	0.375	0.386	0.414	0.423	0.380	0.380	0.392	0.417	0.434	0.388	0.390	0.407	0.412
HG, CO2e (Combined Cycle + margin**)	ton/MWh	0.457	0.412	0.425	0.456	0.465	0.417	0.418	0.431	0.458	0.478	0.427	0.429	0.447	0.453
H3 Slip	ppmvd @ 15% O2	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0	7.0
H3 Slip	lb/hr	24.9	16.5	13.4	12.4	25.7	18.6	18.1	14.7	12.0	26.0	17.4	16.6	13.7	13.0

STG output (kW) estimated by scaling from design case (68 F, evap cooler on, unfired) based on ratio of exhaust energy + duct burner heat input to design case exhaust energy; actual STG output limited to 148 MW with only 1 CTG erating, but maximum duct burner firing rate used for all conditions as worst case for permitting

Margin added for measurement error, off-design conditions and degradation 10.0%

pmbustion turbine performance and emissions data based on GE Energy gas turbine performance simulation runs at site specific conditions; supplemental data calculated as follows

rmulas / Example Calculations

chaust mole flow rate = exhaust mass flow rate / flue gas molecular weight = lb/hr / lb/lb-mole = lb-mole/hr

y exhaust mole flow rate = lb-mole/hr x (1- %H20/100)

ry mole flow rate at 15% O2 = dry lb-mole/hour x (20.9 - % O2 dry)/(20.9/15)

bllutant lb/hr = lb-mole/hr, dry basis, at 15 % O2 x ppmvd @ 15% O2 dry /10⁶ x molecular weight, lb/lb-mole eat Rate (Btu/kWh) = Heat Input Rate (MMBtu/hr) x (10^6 BTU/MMBtu) / Generator Output (MW) / (MW/1000 kW)

Appendix B, Table 2: Annual GHG Emission Calculations – New Combined Cycle Combustion Turbine

Source	Annual Fuel Use ¹ (MMBtu/yr)	Pollutant	Emission Factor ² (kg/MMBtu)	GHG Mass Emissions (tons/yr)	Global Warming Potential ³	CO ₂ e (tons/yr)
		CO ₂	53.02	1,327,623.8	1	1,327,624
Combustion turbine combined		CH ₄	1.00E-03	25.0	25	626
cycle unit	22,716,339	N ₂ O	1.00E-04	2.5	298	746
cycle dine		GHG	Totals	1,327,651		1,328,996
		CO2e	Margin added for meas	surement error =	10%	1,461,896

¹ Annual heat input based on 8760 hours per year of operation of the combustion turbine at an average ambient temperature of 68°F with evaporative cooling on and a duct burner firing at 681.5 MMBtu/hr for 8760 hours per year.

Example Calculations:

Combustion Turbine Fuel = $8760 \text{ hr/yr} \times 1911.6 \text{ MMBtu/hr} = 16,746,049 \text{ MMBtu/yr}$ Duct Burner Fuel = $8760 \text{ hr/yr} \times 681.5 \text{ MMBtu/hr} = 5,970,290 \text{ MMBtu/yr}$ Total Fuel = 16,746,049 + 5,970,290 = 22,716,339 MMBtu/yr

Carbon Dioxide

22,716,339 MMBtu/hr x (53.02 kg CO2/MMBtu) x (2.2046 lb/kg) / (2000 lb/ton) x 1 CO2e/CO2 = 1,327,624 tons CO2e/yr

Methane

22,716,339 MMBtu/hr x (0.001 kg CH4/MMBtu) x (2.2046 lb/kg) / (2000 lb/ton) x 25 CO2e/CH4= 626 tons CO2e/yr

Nitrous Oxides

22,716,339 MMBtu/hr x (0.0001 kg N2O/MMBtu) x (2.2046 lb/kg) / (2000 lb/ton) x 298 CO2e/N2O = 746 tons CO2e/yr

 $^{^{2}}$ CO₂, CH₄ and N₂O emission factors based on Tables C-1 and C-1 of 40 CFR 98.

³ Global warming potential factors based on Table A-1 of 40 CFR 98.

Appendix B, Table 3: Natural Gas Fugitive GHG Emission Calculations

Source	Component	Fluid	Count	Emission Factor ¹	CO2 ²	CH4 ³	Total
	Туре	State		(scf/hr/component)	(tons/yr)	(tons/yr)	(tons/yr)
	Valves		194	0.121	0.093	4.02	
Natural Gas Fugitives	Flanges	Gas/Vapor	161	0.017	0.011	0.47	
	Relief Valve		35	0.193	0.027	1.16	
GHG Mass Based Emissions					0.130	5.64	5.77
Global Warming Potential ⁴					1	25	
CO ₂ e Emissions					0.130	141.0	141.2

 $^{^{1}}$ Emission factors from Table W-1A of 40 CFR Part 98 Mandatory Greenhouse Gas Reporting

0.79% 94.14%

Example Calculations

CO2 Emissions from Valves

194 valves x (0.121 scf gas/hr-valve) x (0.0079 scf CO2/scf gas) / (385.5 scf/lb-mole) x (44 lb CO2/lb-mole) x (8760 hr/yr) / (2000 lb/ton) = 0.093 ton CO2 / year

CH4 Emissions from Valves

194 valves x (0.121 scf gas/hr-valve) x (0.9414 scf CH4/scf gas) / (385.5 scf/lb-mole) x (16 lb CH4/lb-mole) x (8760 hr/yr) / (2000 lb/ton) = 4.017 tons CH4 / year

² CO₂ emissions based on vol % CO₂ in natural gas of

 $^{^3}$ CH $_4$ emissions based on vol % CH $_4$ in natural gas of

⁴ Global warming potential based on Table A-1 of 40 CFR Part 98 Mandatory Greenhouse Gas Reporting

Appendix C BACT Analysis Calculations and Documentation

Appendix C, Table 1 Recent Permit Limits for Natural Gas-Fired Combined Cycle Combustion Turbines > 25 MW Greenhouse Gas Emissions

FACILITY	STATE	PERMIT	PERMIT	STATUS	EQUIPMENT	CONTROL	OUTPUT BASIS	HEAT RATE	INPUT BASIS	CO2e CAP	LIMIT
		STATUS	DATE	OPER?	DETAILS	TECHNOLOGY	LB/MWH	BTU/KWH	LB/MMBTU	TONS/YR	BASIS
RUSSELL CITY	CA	FINAL	2/3/2010	NO	NGCC	EFFICIENT CTGs NG		7,730			VOLUNTARY
PACIFICORP	UT	FINAL	5/4/2011	NO	NGCC	EFFICIENT CTGs & HRSG	950				BACT
ROBINSON POWER	PA	FINAL	6/30/2011	NO	NGCC 1 x GE 7EA + DB	EFFICIENT CTGs				620,000	VOLUNTARY
PALMDALE	CA	FINAL	10/18/2011	NO	NGCC W/SOLAR	EFFICIENT CTGs & SOLAR	774*	7,319	117	1,913,000	BACT
LCRA FERGUSON	TX	FINAL	11/10/2011	NO	NGCC	EFFICIENT CTGs	918	7,720			BACT
KENNECOTT	UT	FINAL	11/22/2011	NO	NGCC	EFFICIENT CTGs		7,642			BACT
PIONEER VALLEY	MA	FINAL	4/12/2012	NO	NGCC MITSUBISHI 501G	EFFICIENT CTGs	895				BACT
CPV VALLEY	NY	DRAFT	7/11/2012	NO	NGCC SIEMENS F	EFFICIENT CTGs	925				BACT
WOODBRIDGE	NJ	FINAL	8/24/2012	NO	NGCC GE 7FA	EFFICIENT CTGs	925	7,605			BACT
GIBSON COUNTY	TN	DRAFT	9/4/2012	NO	NGCC GE 7FA	EFFICIENT CTGs				1,679,459	BACT
CRICKET VALLEY	NY	FINAL	9/27/2012	NO	NGCC GE 7FA	HIGH EFFICIENCY CTGS	913**	7,605	120	3,576,943	BACT
BLACK HILLS	WY	FINAL	9/27/2012	NO	NGCC GE LM6000	EFFICIENT CTGs	1,100			187,318	BACT
HESS NEWARK	NJ	FINAL	10/13/2012	NO	NGCC GE 7FA	EFFICIENT CTGs	887	7,522			BACT
CALPINE DEER PARK	TX	FINAL	11/29/2012	NO	NGCC SIEMENS 501FD2	EFFICIENT CTGs	920	7,730			BACT
CALPINE CHANNEL	TX	FINAL	11/29/2012	NO	NGCC SIEMENS 501FD2	EFFICIENT CTGs	920	7,730			BACT

^{*} Includes solar power component

^{**} Calculated based on heat rate Btu/kWh and emission factor lb/MMBtu

Appendix C - Table 2

CO₂ Pipeline Construction Cost Estimate

Description	041	Basis
Description	Cost ¹	DdSIS
Capital Cost:		
AGI Pipeline - 26" diameter		135 mile pipeline ²
Materials	\$67,055,624	$$64,632 + $1.85 \times L \times (330.5 \times D^2 + 686.7 \times D + 26,960)$
Labor	\$114,212,692	\$341,627 + \$1.85 x L x (343.2 x D ² + 2,074 x D + 170,013)
Miscellaneous	\$48,372,177	\$150,166 + \$1.58 x L x (8,417 x D + 7,234)
Right-of-Way	\$7,304,017	\$48,037 + \$1.20 x L x (577 x D + 29,788)
CO2 Surge Tank	\$1,150,636	
Pipeline Control System	\$110,632	
Total Capital Cost	\$238,205,778	
Annual Operating Cost:		
O&M Cost	\$1,165,320	\$8,632/mile/year
Total Annual Operating Cost	\$1,165,320	
Capital Recovery Factor	0.0944	7% interest rate and 20 year equipment life
Annualized Capital Cost	\$22,486,625	
Total Annual Cost	\$23,651,945	
GHG Emissions Removed	1,314,348	
Cost Effectiveness (\$/ton)	\$18.00	

Length in miles (L) = 135

Diameter in inches (D) = 26

¹ CO₂ transport cost is calculated per pipeline cost equations from *Quality Guidelines for Energy System Studies:* Estimating Carbon Dioxide Transport and Storage Costs, National Energy Technology Laboratory, US Department of Energy, DOE/NETL-2010/1447 (March, 2010).

Appendix C, Table 3

Cost Analysis for Natural Gas Fugitives LDAR Program

Monitoring Cost	\$2.50	per component per quarter
Number of Valves	194	monitored
Number of Flanges	161	not monitored
Number of PRVs	35	monitored
Number of Pumps	0	monitored
Number of Compressors	0	monitored
Total Number Monitored	229	monitored
Total Monitoring Cost	\$2,290	per year
Number of Repairs	110	per year (12% of monitored components per quarter)
Cost of Repairs	\$18,686	per year @ \$200/component (assumed 85% of leaking component need repair; remaining 15% only require minor repair)
Cost to Re-monitor Repairs	\$275	per year
Total Cost of LDAR	\$21,251	per year (monitoring+repair+re-monitoring)
Emission Reduction	97.08	ton/year CO₂e (based on 28M reduction of 75%)
Cost Effectiveness	\$218.90	per ton CO₂e

Example Calculations:

Monitoring cost = $(\$2.50/\text{component/quarter}) \times (229 \text{ components}) \times (4 \text{ quarters/year}) = \$2,290/\text{year}$

Total cost = 2,290/year + 18,686/year + 275/year = 21,251/year

Cost Effectiveness = (\$21,251/year) / (97.08 tons/year CO2e) = \$218.90/ton CO2e